

On the Design of Sliding-Mode Static-Output-Feedback Controllers for Systems With State Delay

X. R. Han, Emilia Fridman, Sarah K. Spurgeon, and Chris Edwards

Abstract—This paper considers the development of sliding-mode-based output-feedback controllers for uncertain systems which are subject to time-varying state delays. A novel method is proposed for design of the switching surface. This method is based on the descriptor approach and leads to a solution in terms of linear matrix inequalities (LMIs). When compared to existing methods (even for systems without delays), the proposed method is efficient and less conservative than other results, giving a feasible solution when the Kimura–Davison conditions are not satisfied. No additional constraints are imposed on the dimensions or structure of the reduced order triple associated with design of the switching surface. The magnitude of the linear gain used to construct the controller is also verified as an appropriate solution to the reachability problem using LMIs. A stability analysis for the full-order time-delay system with discontinuous right-hand side is formulated. This paper facilitates the constructive design of sliding-mode static-output-feedback controllers for a rather general class of time-delay systems. A numerical example from the literature illustrates the efficiency of the proposed method.

Index Terms—Linear matrix inequalities (LMIs), sliding-mode control (SMC), static output feedback (SOF), time delay.

I. INTRODUCTION

SLIDING-MODE control (SMC) [1] is known for its complete robustness to so-called matched uncertainties (which can include time delays that satisfy matching conditions) and disturbances [2]–[4]. The control technique has been applied in many industrial areas [5]–[7]. Many early theoretical developments in SMC assume that all the system states are accessible. In the case where only a subset of states are measurable, which is relevant to a range of practical applications, either output feedback control or the observer-based method are required. Some work has considered implementation of SMC schemes using observers [8]–[10]. In [11], a sliding-mode observer has been shown to give a significant increase in performance in estimation of the unknown variables of a boost converter compared

to a traditional current-mode control strategy. A further interesting strand considers the fast output sampling method [12], [13]. Recently, in [14], a fast sampling method is employed for a discrete systems in the presence of time-varying delays where a sliding-mode controller is designed using linear matrix inequalities (LMIs) combined with a delta-operator approach. However, all these methods require additional computation. The most straightforward approach is to consider the study of SMC via static output feedback (SOF).

One problem of interest in the development of SMC via SOF is the design of the switching surface, which is effectively a reduced order SOF problem for a particular subsystem. Two different methods were proposed to design the sliding surface using eigenvalue assignment and eigenvector techniques in [15] and [16]. A canonical form was provided in [18] via which the SOFSMC design problem is routinely converted to an SOF stabilization problem. As stated in [20], all previous-reported methods for the existence problem are, in fact, equivalent to a particular SOF problem. The solution to the general SOF problem, even for linear time-invariant systems, is still open.

LMI methods have been considered within the context of sliding-mode controller design. For example, [21] and [22] presented LMIs methods to design static sliding-mode output-feedback controllers and [23] presented a necessary and sufficient condition to solve the existence problem in terms of LMIs for linear uncertain systems.

It is important to note that all the work described above does not consider an existence problem involving delay and many practical problems include such effects [24], [25]. In [26], the problem of the development of sliding-mode controllers for operation in the presence of single or multiple, constant or time-varying state delays has been solved. This uses the usual regular form method of solution and the uncertainty is assumed to be matched, where matched describes that uncertainty class which is implicit in the range of the input channels and will be rejected by an appropriately designed SMC strategy, although it is important to note that full state availability is assumed. This problem has also been considered in [27] where a class of uncertain time delay systems with multiple fixed delays in the system states is considered. This paper considers unmatched and time-varying parameter uncertainties, together with matched and bounded external disturbances, but again, full state information is assumed to be available to the controller. In [28], Lyapunov functionals were for the first time introduced for the analysis of time-varying delay. In [29], a descriptor

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85 approach to stability and control of linear systems with time-
86 varying delays, which is based on the Lyapunov–Krasovskii
87 techniques, was combined with results on the SMC of such
88 systems. The systems under consideration were subjected to
89 norm-bounded uncertainties and uncertain bounded delays and
90 the solution given in terms of LMIs. Reference [30] develops
91 a SMC synthesis for a class of uncertain time-delay systems
92 with nonlinear disturbances and unknown delay values whose
93 unperturbed dynamics is linear. The synthesis was based on a
94 new delay-dependent stability criterion. The controller is found
95 to be robust against sufficiently small delay variations and
96 external disturbances.

97 It is important to emphasize that much of the aforementioned
98 literature on SMC of time-delay systems assumes full-state
99 feedback. Reference [31] considered SMC of systems with
100 time-varying delay. This paper considered a solution via LMIs
101 for the existence problem. The current paper extends this contri-
102 bution to consider the solution of the existence and reachability
103 problems. Specifically, the selection of parameters for stability
104 of the full-order closed-loop system are obtained via LMIs. In
105 this paper, a compensator-based design problem is considered
106 using the proposed SOF approach. Example from the literature
107 illustrates the efficiency of the method. In Section II, the
108 problem formulation is described. The existence problem is
109 formulated in Section III. In Sections IV and V, stability of
110 the full-order closed-loop system is derived via LMIs, and the
111 reachability problem is presented. Compensator-based design
112 is demonstrated in Section VI.

113 II. PROBLEM FORMULATION

114 Consider an uncertain time-delay system

$$\begin{aligned} \dot{z}(t) &= Az(t) + A_d z(t - \tau(t)) + B(u(t) + \xi(t, z, u)) \\ y(t) &= Cz(t) \end{aligned} \quad (1)$$

115 where $z \in \mathcal{R}^n$, $u \in \mathcal{R}^m$, and $y \in \mathcal{R}^p$ with $m \leq p \leq n$. The
116 time-varying delay $\tau(t)$ is supposed to be bounded $0 \leq \tau(t) \leq$
117 h , and it may be either slowly varying (i.e., differentiable
118 delay with $\dot{\tau}(t) \leq d < 1$) or fast varying (piecewise continuous
119 delay). Assume that the nominal linear system (A, A_d, B, C)
120 is known and that the input and output matrices B and C are
121 both of full rank. The unknown function $\xi: \mathcal{R}_+ \times \mathcal{R}^n \times \mathcal{R}^m \rightarrow$
122 \mathcal{R}^n , which represents the system nonlinearities plus any model
123 uncertainties, is assumed to satisfy the matching condition and

$$\|\xi(t, z, u)\| < k_1 \|u\| + a(t, y) \quad (2)$$

124 for some known function $a: \mathcal{R}_+ \times \mathcal{R}^p \rightarrow \mathcal{R}_+$ and positive
125 constant $k_1 < 1$. It can be shown that if $\text{rank}(CB) = m$, there
126 exists a coordinate system in which the system (A, A_d, B, C)
127 has the structure

$$\begin{aligned} A &= \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} & A_d &= \begin{bmatrix} A_{d11} & A_{d12} \\ A_{d21} & A_{d22} \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ B_2 \end{bmatrix} & C &= [0 \quad T] \end{aligned} \quad (3)$$

where $B_2 \in \mathcal{R}^{m \times m}$ is nonsingular and $T \in \mathcal{R}^{p \times p}$ is orthogo- 128
nal. The system can be represented as 129

$$\begin{aligned} \dot{z}_1(t) &= A_{11}z_1(t) + A_{d11}z_1(t - \tau(t)) \\ &\quad + A_{12}z_2(t) + A_{d12}z_2(t - \tau(t)) \\ \dot{z}_2(t) &= \sum_{i=1}^2 (A_{2i}z_i(t) + A_{d2i}z_i(t - \tau(t))) \\ &\quad + B_2(u(t) + \xi(t, z, u)) \\ y(t) &= Cz(t). \end{aligned} \quad (4)$$

Consider the following switching function: 130 AQ1

$$S = \{z(t) \in \mathcal{R}^n : FCz(t) = 0\} \quad (5)$$

for some selected matrix $F \in \mathcal{R}^{m \times p}$ where by design 131
 $\det(FCB) \neq 0$. Let 132

$$[F_1 \quad F_2] = FT \quad (6)$$

where $F_1 \in \mathcal{R}^{p-m}$ and $F_2 \in \mathcal{R}^m$. As a result 133

$$FC = [F_1 C_1 \quad F_2] \quad (7)$$

where 134

$$C_1 = [0_{(p-m) \times (n-p)} \quad I_{(p-m)}]. \quad (8)$$

Therefore, $FCB = F_2 B_2$ and the square matrix F_2 is 135
nonsingular. By assumption, the uncertainty is matched, and 136
therefore the sliding motion is independent of the uncertainty 137
represented by $\xi(\cdot)$. In addition, because the canonical form in 138
(3), where it is necessary that the pair (A_{11}, A_{12}) is controllable 139
and (A_{11}, C_1) is observable, can be viewed as a special case 140
of the regular form normally used in sliding-mode controller 141
design, the switching function can also be expressed as 142

$$s(t) = z_2(t) + KC_1 z_1(t) \quad (9)$$

where $K \in \mathcal{R}^{m \times (p-m)}$ and is defined as $K = F_2^{-1} F_1$. 143

Then, a simple SMC law, depending on the output informa- 144
tion $Fy(t)$ can be defined by 145

$$u(t) = -\gamma Fy(t) - v(t) \quad (10)$$

where 146

$$v(t) = \begin{cases} \rho(t, y) \frac{Fy(t)}{\|Fy(t)\|}, & \text{if } Fy(t) \neq 0 \\ 0, & \text{otherwise} \end{cases} \quad (11)$$

where $\rho(t, y)$ is some positive scalar function of the outputs 147

$$\rho(t, y) = (k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2) / (1 - k_1)$$

where γ and γ_2 are positive design scalars [18]. The closed- 148
loops system (4) and (10) can be described by the following 149
equations: 150

$$\begin{aligned} \dot{z}_1(t) &= (A_{11} - A_{12}KC_1)z_1(t) \\ &\quad + (A_{d11} - A_{d12}KC_1)z_1(t - \tau(t)) \end{aligned}$$

$$\begin{aligned}
\dot{s}(t) &= (A_{21} - \gamma B_2 K C_1) z_1(t) \\
&+ (A_{d21} - \gamma B_2 K C_1) z_1(t - \tau(t)) \\
&+ (A_{22} - \gamma B_2) z_2(t) + (A_{d22} - \gamma B_2) z_2(t - \tau(t)) \\
&+ B_2 (\xi(t, z, u) - v(t)) \\
y(t) &= C z(t). \tag{12}
\end{aligned}$$

151 III. EXISTENCE PROBLEM

152 On the sliding manifold $s(t) = 0$, it is well known [19] that
 153 the reduced-order sliding motion is governed by a free motion
 154 with system matrix

$$\begin{aligned}
\dot{z}_1(t) &= (A_{11} - A_{12} K C_1) z_1(t) \\
&+ (A_{d11} - A_{d12} K C_1) z_1(t - \tau(t)). \tag{13}
\end{aligned}$$

155 Consider a Lyapunov–Krasovskii functional

$$\begin{aligned}
V(t) &= z_1^T(t) P z_1(t) + \int_{t-h}^t z_1^T(s) E z_1(s) ds \\
&+ \int_{t-\tau(t)}^t z_1^T(s) S z_1(s) ds \\
&+ h \int_{-h}^0 \int_{t+\theta}^t z_1^T(s) R \dot{z}_1(s) ds d\theta \tag{14}
\end{aligned}$$

156 where the symmetric matrices $P > 0$ and $E, S, R \geq 0$.
 157 The condition $\dot{V}(t) < 0$ guarantees asymptotic stability of
 158 the reduced order system as in [32]. Differentiating $V(t)$
 159 along (13)

$$\begin{aligned}
\dot{V}(t) &= 2z_1^T(t) P \dot{z}_1(t) + h^2 \dot{z}_1^T(t) R \dot{z}_1(t) \\
&- h \int_{t-h}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds + z_1^T(t) (E + S) z_1(t) \\
&- z_1^T(t-h) E z_1(t-h) \\
&- (1 - \dot{\tau}(t)) z_1^T(t - \tau(t)) S z_1(t - \tau(t)). \tag{15}
\end{aligned}$$

160 Further using the identity

$$\begin{aligned}
-h \int_{t-h}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds &= -h \int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) R \dot{z}_1(s) ds \\
&- h \int_{t-\tau(t)}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds \tag{16}
\end{aligned}$$

161 and applying Jensen's inequality

$$\int_{t-\tau(t)}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds \geq \frac{1}{h} \int_{t-\tau(t)}^t \dot{z}_1^T(s) ds R \int_{t-\tau(t)}^t \dot{z}_1(s) ds \tag{17}$$

$$\int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) R \dot{z}_1(s) ds \geq \frac{1}{h} \int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) ds R \int_{t-h}^{t-\tau(t)} \dot{z}_1(s) ds. \tag{18}$$

Then,

$$\begin{aligned}
\dot{V}(t) &\leq 2z_1^T(t) P \dot{z}_1(t) + h^2 \dot{z}_1^T(t) R \dot{z}_1(t) \\
&- (z_1(t) - z_1(t - \tau(t)))^T R (z_1(t) - z_1(t - \tau(t))) \\
&- (z_1(t - \tau(t)) - z_1(t - h))^T \\
&\times R (z_1(t - \tau(t)) - z_1(t - h)) \\
&+ z_1^T(t) (E + S) z_1(t) - z_1^T(t - h) E z_1(t - h) \\
&- (1 - d) z_1^T(t - \tau(t)) S z_1(t - \tau(t)). \tag{19}
\end{aligned}$$

Using the descriptor method as in [33] and the free-weighting
 163 matrices technique from [34], the right-hand side of the
 164 expression
 165

$$\begin{aligned}
0 &\equiv 2(z_1^T(t) P_2^T + \dot{z}_1^T(t) P_3^T) \\
&\times [-\dot{z}_1(t) + (A_{11} - A_{12} K C_1) z_1(t) \\
&+ (A_{d11} - A_{d12} K C_1) z_1(t - \tau(t))] \tag{20}
\end{aligned}$$

with matrix parameters $P_2, P_3 = \epsilon P_2 \in \mathcal{R}^{n-m}$ is
 166 added into the right-hand side of (19). Setting $\eta(t) =$
 167 $\text{col}\{z_1(t), \dot{z}_1(t), z_1(t - h), z_1(t - \tau(t))\}$, it follows that
 168

$$\dot{V}(t) \leq \eta^T(t) \Theta \eta(t) \leq 0 \tag{21}$$

if the matrix inequality

$$\Theta = \begin{bmatrix} \theta_{11} & \theta_{12} & 0 & \theta_{14} \\ * & \theta_{22} & 0 & \theta_{24} \\ * & * & \theta_{33} & \theta_{34} \\ * & * & * & \theta_{44} \end{bmatrix} < 0 \tag{22}$$

is feasible, where

$$\begin{aligned}
\theta_{11} &= P_2^T (A_{11} - A_{12} K C_1) \\
&+ (A_{11} - A_{12} K C_1)^T P_2 + E + S - R \\
\theta_{12} &= P - P_2^T + \epsilon (A_{11} - A_{12} K C_1)^T P_2 \\
\theta_{14} &= P_2^T (A_{d11} - A_{d12} K C_1) + R \\
\theta_{22} &= -\epsilon P_2 - \epsilon P_2^T + h^2 R \\
\theta_{24} &= \epsilon P_2^T (A_{d11} - A_{d12} K C_1) \\
\theta_{33} &= -(E + R) \\
\theta_{34} &= R \\
\theta_{44} &= -2R - (1 - d)S. \tag{23}
\end{aligned}$$

Multiplying matrix Θ from the right and the left by
 171 $\text{diag}\{P_2^{-1}, P_2^{-1}, P_2^{-1}, P_2^{-1}\}$ and its transpose, respectively, and
 172 denoting
 173

$$\begin{aligned}
Q_2 &= P_2^{-1} \quad \hat{P} = Q_2^T P Q_2 \quad \hat{R} = Q_2^T R Q_2 \\
\hat{E} &= Q_2^T E Q_2 \quad \hat{S} = Q_2^T S Q_2
\end{aligned}$$

174 it follows $\Theta < 0 \Leftrightarrow \hat{\Theta} < 0$, where

$$\hat{\Theta} = \begin{bmatrix} \hat{\theta}_{11} & \hat{\theta}_{12} & 0 & \hat{\theta}_{14} \\ * & \hat{\theta}_{22} & 0 & \hat{\theta}_{24} \\ * & * & \hat{\theta}_{33} & \hat{\theta}_{34} \\ * & * & * & \hat{\theta}_{44} \end{bmatrix} < 0 \quad (24)$$

$$\begin{aligned} \hat{\theta}_{11} &= (A_{11} - A_{12}KC_1)Q_2 \\ &\quad + Q_2^T(A_{11} - A_{12}KC_1)^T + \hat{E} + \hat{S} - \hat{R} \\ \hat{\theta}_{12} &= \hat{P} - Q_2 + \epsilon Q_2^T(A_{11} - A_{12}KC_1)^T \\ \hat{\theta}_{14} &= (A_{d11} - A_{d12}KC_1)Q_2 + \hat{R} \\ \hat{\theta}_{22} &= -\epsilon Q_2 - \epsilon Q_2^T + h^2 \hat{R} \\ \hat{\theta}_{24} &= \epsilon(A_{d11} - A_{d12}KC_1)Q_2 \\ \hat{\theta}_{33} &= -\hat{E} - \hat{R} \\ \hat{\theta}_{34} &= \hat{R} \\ \hat{\theta}_{44} &= -2\hat{R} - (1-d)\hat{S}. \end{aligned} \quad (25)$$

175 Define the variable Q_2 in the following form:

$$Q_2 = \begin{bmatrix} Q_{11} & Q_{12} \\ Q_{22}\mathcal{M} & \delta Q_{22} \end{bmatrix} \quad (26)$$

176 where Q_{22} is a $(p-m) \times (p-m)$ matrix, and $\mathcal{M} \in$
177 $\mathcal{R}^{(p-m) \times (n-p)}$ and $\delta \in \mathcal{R}$ are *a priori* selected tuning param-
178 ters. It follows that

$$KC_1Q_2 = [KQ_{22}\mathcal{M} \quad \delta KQ_{22}].$$

179 Defining

$$Y = KQ_{22}$$

180 it follows that

$$KC_1Q_2 = [Y\mathcal{M} \quad \delta Y]. \quad (27)$$

181 To construct K , substitute (27) into (25) to yield

$$\begin{aligned} \hat{\theta}_{11} &= A_{11}Q_2 - A_{12}[Y \quad \delta Y] + Q_2^T A_{11}^T \\ &\quad - [Y\mathcal{M} \quad \delta Y]^T A_{12}^T + \hat{E} + \hat{S} - \hat{R} \\ \hat{\theta}_{12} &= \hat{P} - Q_2 + \epsilon Q_2^T A_{11}^T - \epsilon [Y\mathcal{M} \quad \delta Y]^T A_{12}^T \\ \hat{\theta}_{14} &= A_{d11}Q_2 - A_{d12}[Y\mathcal{M} \quad \delta Y] + \hat{R} \\ \hat{\theta}_{22} &= -\epsilon Q_2 - \epsilon Q_2^T + h^2 \hat{R} \\ \hat{\theta}_{24} &= \epsilon A_{d11}Q_2 - \epsilon A_{d12}[Y\mathcal{M} \quad \delta Y] \\ \hat{\theta}_{33} &= -\hat{E} - \hat{R} \\ \hat{\theta}_{34} &= \hat{R} \\ \hat{\theta}_{44} &= -2\hat{R} - (1-d)\hat{S} \end{aligned} \quad (28)$$

182 with the tuning parameters δ , ϵ , and \mathcal{M} . The following Lemma
183 may now be stated.

184 *Lemma 1:* Given *a priori* selected tuning parameters ϵ , δ ,
185 and $\mathcal{M} \in \mathcal{R}^{(p-m) \times (n-p)}$, then (24) is an LMI in the decision
186 variables $\hat{P} > 0$, $\hat{E} \geq 0$, $\hat{S} \geq 0$, $\hat{R} \geq 0$ and matrices $Q_{22} \in$

$\mathcal{R}^{(p-m) \times (p-m)}$, $Q_{11} \in \mathcal{R}^{(n-p) \times (n-p)}$, $Q_{12} \in \mathcal{R}^{(n-p) \times (p-m)}$, 187
 $Y \in \mathcal{R}^{m \times (p-m)}$. If a solution to (24) exists, which may be read- 188
ily obtained from available LMI tools, then the reduced order 189
system (13) is asymptotically stable for all differentiable delays 190
 $0 \leq \tau(t) \leq h$, $\dot{\tau}(t) \leq d < 1$. Moreover, (13) is asymptotically 191
stable for all piecewise-continuous delays $0 \leq \tau(t) \leq h$, if the 192
LMI (24) is feasible with $\hat{S} = 0$. 193

Remark: The proposed method is suitable for SOF sliding- 194
mode controller design where Kimura–Davison conditions, 195
written as $n \leq m + p - 1$, are not satisfied as in [20]–[22]. No 196
constraints are imposed on the dimensions of the reduced-order 197
triple A_{11} , A_{12} , C_1 . This represents a constructive and efficient 198
approach to output-feedback-based design for a relatively broad 199
class of systems, which is less conservative than existing results 200
[20]–[22]; an example to illustrate the advantages of the method 201
for systems without time-delay is presented in [17]. 202

IV. STABILITY OF THE FULL-ORDER CLOSED-LOOP SYSTEM

203
204 It can be shown [18] that there exist a coordinate system in 205
which the system triple $(\bar{A}, \bar{A}_d, \bar{B}, \bar{F}\bar{C})$ has the property that 206

$$\begin{aligned} \bar{A} &= \begin{bmatrix} \bar{A}_{11} & \bar{A}_{12} \\ \bar{A}_{21} & \bar{A}_{22} \end{bmatrix} & \bar{A}_d &= \begin{bmatrix} \bar{A}_{d11} & \bar{A}_{d12} \\ \bar{A}_{d21} & \bar{A}_{d22} \end{bmatrix} \\ \bar{B} &= \begin{bmatrix} 0 \\ B_2 \end{bmatrix} & \bar{F}\bar{C} &= [0 \quad F_2] \end{aligned} \quad (29)$$

where $\bar{A}_{11} = A_{11} - A_{12}KC_1$ and $\bar{A}_{d11} = A_{d11} - A_{d12}KC_1$ 207
and F_2 is a design parameter. Let \bar{P} be a symmetric positive 208
definite matrix partitioned conformably with the matrices in 209
(29) so that 210

$$\bar{P} = \begin{bmatrix} \bar{P}_1 & 0 \\ 0 & \bar{P}_2 \end{bmatrix} \quad (30)$$

then the matrix \bar{P} satisfies the structural constraint 211

$$\bar{P}\bar{B} = \bar{C}^T F^T \quad (31)$$

if the design matrix $F_2 = B_2^T \bar{P}_2$. The matrix \bar{P} can be shown 212
to be a Lyapunov matrix for 213

$$\begin{aligned} \bar{A}_0 &= \bar{A} - \gamma \bar{B} \bar{F} \bar{C} \\ &= \bar{A} - \gamma \bar{B} [0 \quad F_2] \end{aligned} \quad (32)$$

for sufficiently large γ [18]. In the new coordinate system, the 214
uncertain system (1) can be written as 215

$$\dot{z}(t) = \bar{A}z(t) + \bar{A}_d z(t - \tau) + \bar{B}(u(t) + \xi(t, z, u)). \quad (33)$$

The closed-loop system will have the form 216

$$\dot{z}(t) = \bar{A}_0 z(t) + \bar{A}_d z(t - \tau) + \bar{B}(\xi(t, y_t) - v_y(t)). \quad (34)$$

For large enough $\gamma > 0$, these conditions are delay independent 217
with respect to the delay in z_2 .} However, for derivation of 218
this condition using Lyapunov–Krasovskii techniques, it is 219
necessary to consider the case where $\dot{\tau} \leq d < 1$. A stability 220

221 condition for the full-order closed-loop system can be derived
222 using the following Lyapunov–Krasovskii functional:

$$V(t) = z^T(t)\bar{P}z(t) + \int_{t-h}^t z^T(s)\bar{E}z(s)ds + \int_{t-\tau(t)}^t z^T(s)\bar{S}z(s)ds + h \int_{-h}^0 \int_{t+\theta}^t z^T(s)\bar{R}\dot{z}(s)dsd\theta \quad (35)$$

223 where the matrix \bar{E} , $\bar{S} \geq 0$ and $\bar{R} = \begin{bmatrix} \bar{R}_1 & 0 \\ 0 & 0 \end{bmatrix} \geq 0$ as it is
224 desired to determine a stability condition for the time delay
225 system which is delay independent with respect to delay in
226 $z_2(t)$. Differentiating $V(t)$ along the closed-loop trajectories

$$\begin{aligned} \dot{V}(t) &\leq 2z^T(t)\bar{P}\dot{z}(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) \\ &\quad - (z(t) - z(t - \tau(t)))^T \bar{R} (z(t) - (t - \tau(t))) \\ &\quad - (z(t - \tau(t)) - z(t - h))^T \\ &\quad \times \bar{R} (z(t - \tau(t)) - z(t - h)) \\ &\quad + z^T(t)(\bar{E} + \bar{S})z(t) - z^T(t - h)\bar{E}z(t - h) \\ &\quad - (1 - d)z^T(t - \tau(t))\bar{S}z(t - \tau(t)). \end{aligned} \quad (36)$$

227 Substitute the right-hand side of (34) into (36). Setting $\varsigma(t) =$
228 $col\{z(t), z(t - h), z(t - \tau(t))\}$, it follows that

$$\dot{V}(t) \leq \varsigma(t)^T \Phi_h \varsigma(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) + 2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) < 0 \quad (37)$$

229 is satisfied if $\varsigma^T(t)\Phi_h\varsigma(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) < 0$ and
230 $2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) < 0$, where

$$\Phi_h = \begin{bmatrix} \phi_{11} - \bar{R} & 0 & \bar{P}\bar{A}_d + \bar{R} \\ * & -(\bar{E} + \bar{R}) & \bar{R} \\ * & * & -2\bar{R} - (1 - d)\bar{S} \end{bmatrix} \quad (38)$$

231 with

$$\phi_{11} = \bar{A}_0^T \bar{P} + \bar{P} \bar{A}_0 + \bar{S} + \bar{E}. \quad (39)$$

232 Setting $\varrho(t) = col\{z(t), z(t - h), z(t - \tau), \xi(t, z, u) - v(t)\}$

$$\begin{aligned} &h^2\dot{z}^T(t)\bar{R}\dot{z}(t) \\ &= h^2 \left[z^T(t)\bar{A}_0 + z^T(t - \tau)\bar{A}_d^T \right. \\ &\quad \left. + (\xi(t, z, u) - v(t))^T \bar{B}^T \right] \bar{R} \\ &\quad \times [\bar{A}_0 z + \bar{A}_d z(t - \tau) + \bar{B}(\xi(t, z, u) - v(t))] \\ &= \varrho^T(t) \begin{bmatrix} \bar{A}_0^T \\ 0_{n_T} \\ \bar{A}_d^T \\ \bar{B}^T \end{bmatrix} \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} h^2 \bar{R}_1 \\ &\quad \times \begin{bmatrix} I_{(n-qm)} \\ 0 \end{bmatrix}^T \begin{bmatrix} \bar{A}_0^T \\ 0_{n_T} \\ \bar{A}_d^T \\ \bar{B}^T \end{bmatrix}^T \varrho(t). \end{aligned} \quad (40)$$

Using the Schur complement, $\xi^T(t)\Phi_h\xi(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) < 0$
holds if 233 234

$$\begin{bmatrix} h\bar{A}_0^T \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} \bar{R}_1 \\ \Phi_h \\ h\bar{A}_d^T \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} \bar{R}_1 \\ *** \\ -\bar{R}_1 \end{bmatrix} < 0. \quad (41)$$

Inequality (41) is an LMI in the decision variables $\bar{P}_1 > 0$, $\bar{E} \geq 0$,
235 0 , $\bar{S} \geq 0$ and $\bar{R}_1 \geq 0$. Equation (37) is valid if (41) is satisfied
236 and given 237

$$\begin{aligned} &2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) \\ &= 2y^T F^T (\xi(t, z, u) - v(t)) \\ &\leq -2y^T F^T v(t) + 2\|Fy(t)\| \|\xi(t, z, u)\| \\ &= -2\rho(t, y) \|Fy(t)\| + 2\|Fy(t)\| \|\xi(t, z, u)\| \\ &< -2\|Fy(t)\| (\rho(t, y) - k_1 \|u(t)\| - \alpha(t, y)). \end{aligned} \quad (42)$$

238 However, by definition 238

$$\rho(t, y) = (k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2) / (1 - k_1)$$

239 and so by rearranging 239

$$\begin{aligned} \rho(t, y) &= k_1 \rho(t, y) + k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2 \\ &\geq k_1 (\|v(t)\| + \gamma \|Fy(t)\|) + \alpha(t, y) + \gamma_2 \\ &\geq k_1 \|u(t)\| + \alpha(t, y) + \gamma_2. \end{aligned} \quad (43)$$

240 From (37), if (41) is valid, then from (42) and (43), 240

$$\dot{V}(t) < -2\gamma_2 \|Fy(t)\| < 0 \quad \text{if } z(t) \neq 0 \quad (44)$$

241 and therefore the system is asymptotically stable. 241

Lemma 2: Given large enough γ , let there exist $n \times n$ ma-
242 trices $\bar{P}_1 > 0$, $\bar{E} \geq 0$, $\bar{S} \geq 0$, $\bar{R}_1 \geq 0$ from the LMI solver
243 such that LMI (41) holds. Given that the design parameters
244 k_1 , $\alpha(t, y)$, γ_2 , and \bar{P}_2 have been selected so that (44) holds,
245 the closed-loop system (33) is asymptotically stable for all
246 differentiable delays $0 \leq \tau(t) \leq h$, $\dot{\tau}(t) \leq d \leq 1$. 247

V. FINITE-TIME REACHABILITY TO THE SLIDING MANIFOLD 248 249

Corollary: An ideal sliding motion takes place on the sur-
250 face S if 251

$$\|B_2^{-1} \bar{A}_0^L z(t)\| + \|B_2^{-1} \bar{A}_d^L z(t - \tau)\| < \gamma_2 - \eta \quad (45)$$

252 where the matrices \bar{A}_0^L and \bar{A}_d^L represent the last m rows of
253 \bar{A}_0 and \bar{A}_d , respectively, and η is a small scalar satisfying
254 $0 < \eta < \gamma_2$. 254

Proof: 255

$$\dot{s}(t) = F\bar{C}\bar{A}_0\bar{z}(t) + F\bar{C}\bar{A}_d\bar{z}(t - \tau) + F_2 B_2 (\xi(t, z, u) - v(t)). \quad (46)$$

256 Let $V_c : \mathcal{R}^m \rightarrow \mathcal{R}$ be defined by

$$V_c(s) = s^T(t) (F_2^{-1})^T \bar{P}_2 F_2^{-1} s(t). \quad (47)$$

257 Then, using the fact that $F_2^T = \bar{P}_2 B_2$ it follows that

$$\begin{aligned} (F_2^{-1})^T \bar{P}_2 F_2^{-1} F \bar{C} \bar{A}_0 &= B_2^{-1} \bar{A}_0^L \\ (F_2^{-1})^T \bar{P}_2 F_2^{-1} F \bar{C} \bar{A}_d &= B_2^{-1} \bar{A}_d^L. \end{aligned} \quad (48)$$

258 Then, it can be verified that

$$\begin{aligned} \dot{V}_c &= 2s^T(t) B_2^{-1} \bar{A}_0^L z(t) + 2s^T(t) B_2^{-1} \bar{A}_d^L z(t - \tau) \\ &\quad + 2s^T(t) (\xi(t, z, u) - v(t)) \\ &\leq 2 \|s(t)\| \|B_2^{-1} \bar{A}_0^L z(t)\| + 2 \|s(t)\| \\ &\quad \times \|B_2^{-1} \bar{A}_d^L z(t - \tau)\| - 2\gamma_2 \|s(t)\| \\ &< -2\eta \|s(t)\| \end{aligned} \quad (49)$$

259 if $z(t)$ and $z(t - \tau) \in \Omega$. It follows that there exists a t_0 such
260 that $z(t)$ and $z(t - \tau) \in \Omega$ for all $t > t_0$. Consequently, (49)
261 holds for all $t > t_0$. A sliding motion will thus be attained in
262 finite time.

263 *Example 1:* The following model of a liquid monopropel-
264 lant rocket motor has been considered in [35]. It is assumed
265 that the variable $\kappa = 0.8$ in this case, where $A_d(1, 1) = -\kappa$
266 and $A(1, 1) = \kappa - 1$. The outputs have been chosen to be the
267 second and fourth states so that in (1)

$$\begin{aligned} A &= \begin{bmatrix} -0.2 & 0 & 0 & 0 \\ 0 & 0 & 0 & -1 \\ -1 & 0 & -1 & 1 \\ 0 & 1 & 1 & 0 \end{bmatrix} & A_d &= \begin{bmatrix} -0.8 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ 1 \\ 0 \\ 0 \end{bmatrix} & C &= \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}. \end{aligned} \quad (50)$$

268 Clearly, the Kimura–Davison conditions are not met. Here,
269 the rate at which the delay varies with time has been examined
270 with $d = 0$, a slow varying delay. The gain from the LMI tool
271 solver with $\delta = 50$, $\epsilon = 0.5$, $h = 0.45s$, $\bar{P}_2 = 1$, $\gamma = 2.9$, and
272 $M = [5 \ 0.2]$, yields $F = [-1 \ -2.0754]$. The poles of the
273 sliding-mode dynamics are $\{-2.67, -0.4, -0.2\}$. A simu-
274 lation was performed with the initial state values $[1 \ 1 \ 1 \ 1]$.
275 As shown, the LMI solver gave a feasible result for stability
276 for $h \leq 0.45s$, but in simulation, the closed-loop system only
277 became unstable for $h \geq 1s$ (Fig. 1); this is due to the conserv-
278 ativeness of the method. The LMI solver gave feasible closed-
279 loop stability results for controller gain $\gamma \geq 2.9$ while a choice
280 of $\gamma \geq 1$ in simulation was able to stabilize the system with the
281 compensation of longer settling time. The switching function
282 for $h = 0.45s$, $\gamma = 2.9$ is shown in Fig. 2.

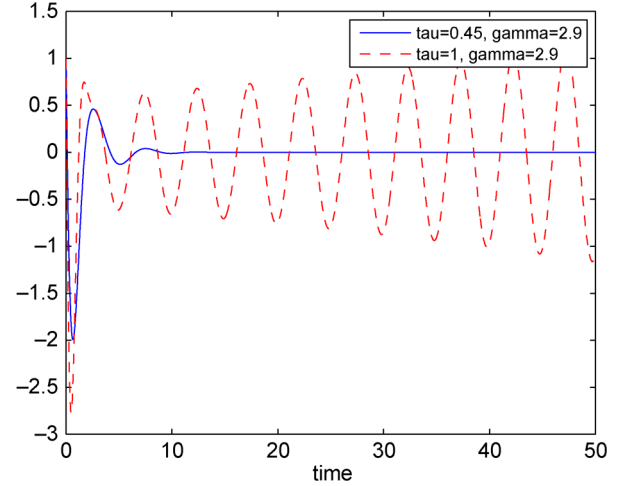


Fig. 1. Output against time.

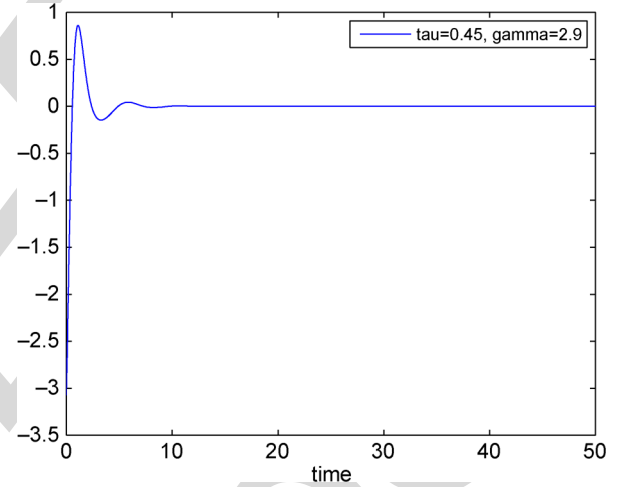


Fig. 2. Switching function.

VI. COMPENSATOR-BASED EXISTENCE PROBLEM 283

For certain system triples (A_{11}, A_{12}, C_1) , LMI (24) is known
284 to be infeasible. In this case, consider a dynamic compensator
285 similar to that of El-Khazali and DeCarlo [36]
286

$$\dot{z}_c(t) = H z_c(t) + D y(t) \quad (51)$$

where the matrices $H \in \mathcal{R}^{q \times q}$ and $D \in \mathcal{R}^{q \times p}$ are to be deter-
287 mined. Define a new hyperplane in the augmented state space,
288 formed from the plant and compensator state spaces, as
289

$$S_c = \{(z(t), z_c(t)) \in \mathcal{R}^{n+q} : F_c z_c(t) + F z(t) = 0\} \quad (52)$$

where $F_c \in \mathcal{R}^{m \times q}$ and $F \in \mathcal{R}^{m \times p}$. Define $D_1 \in \mathcal{R}^{q \times (p-m)}$
290 and $D_2 \in \mathcal{R}^{q \times m}$ as
291

$$[D_1 \ D_2] = DT \quad (53)$$

then the compensator can be written as
292

$$\dot{z}_c(t) = H z_c(t) + D_1 C_1 z_1(t) + D_2 z_2(t) \quad (54)$$

293 where C_1 is defined in (8). The sliding motion, obtained by
294 eliminating the coordinates $z_2(t)$, can be written as

$$\begin{aligned} \dot{z}_1(t) &= (A_{11} - A_{12}KC_1)z_1(t) - A_{12}K_c z_c(t) \\ &\quad + (A_{d11} - A_{d12}KC_1)z_1(t - \tau) - A_{d12}K_c z_c(t - \tau) \\ \dot{z}_c(t) &= (D_1 - D_2K)C_1 z_1(t) + (H - D_2K_c)z_c(t) \end{aligned} \quad (55)$$

295 where $K = F_2^{-1}F_1$ and $K_c = F_2^{-1}F_c$, then similar to [37], the
296 design problem becomes one of selecting a compensator, re-
297 presented by the matrices D_1 , D_2 , and H , and a hyperplane,
298 represented by the matrices K and K_c , so that the system

$$\begin{aligned} \begin{bmatrix} \dot{z}_1(t) \\ \dot{z}_c(t) \end{bmatrix} &= \underbrace{\begin{bmatrix} A_{11} - A_{12}KC_1 & -A_{12}K_c \\ (D_1 - D_2K)C_1 & H - D_2K_c \end{bmatrix}}_{A_c} \begin{bmatrix} z_1(t) \\ z_c(t) \end{bmatrix} \\ &\quad + \underbrace{\begin{bmatrix} A_{d11} - A_{d12}KC_1 & -A_{d12}K_c \\ 0 & 0 \end{bmatrix}}_{A_{cd}} \begin{bmatrix} z_1(t - \tau) \\ z_c(t - \tau) \end{bmatrix} \end{aligned} \quad (56)$$

299 is stable. To obtain the compensator gains this problem can be
300 shown to be a new output-feedback problem with

$$\begin{aligned} A_c &= \underbrace{\begin{bmatrix} A_{11} & 0 \\ 0 & 0 \end{bmatrix}}_{A_q} - \underbrace{\begin{bmatrix} A_{12} & 0 \\ D_2 & -I_q \end{bmatrix}}_{B_q} \underbrace{\begin{bmatrix} K & K_c \\ D_1 & H \end{bmatrix}}_{K_q} \underbrace{\begin{bmatrix} C_1 & 0 \\ 0 & I_q \end{bmatrix}}_{C_q} \\ A_{cd} &= \underbrace{\begin{bmatrix} A_{d11} & 0 \\ 0 & 0 \end{bmatrix}}_{A_{qd}} - \underbrace{\begin{bmatrix} A_{d12} & 0 \\ 0 & 0 \end{bmatrix}}_{B_{qd}} \underbrace{\begin{bmatrix} K & K_c \\ D_1 & H \end{bmatrix}}_{K_q} \underbrace{\begin{bmatrix} C_1 & 0 \\ 0 & I_q \end{bmatrix}}_{C_q}. \end{aligned} \quad (57)$$

301 The existence problem represented by system (56), where A_c
302 and A_{cd} are partitioned as in (57) and D_2 is a tuning parameter,
303 can be solved as for the noncompensated case (13). Similarly
304 to (27),

$$\begin{aligned} K_q C_q Q_2 &= K_q \begin{bmatrix} 0_{(p-m+q) \times (n-p)} & I_{p-m+q} \end{bmatrix} \begin{bmatrix} Q_{11} & Q_{12} \\ Q_{22} \mathcal{M} & \delta Q_{22} \end{bmatrix} \\ &= [K_q Q_{22} \mathcal{M} \quad \delta K_q Q_{22}] \\ &= [Y \mathcal{M} \quad \delta Y] \end{aligned} \quad (58)$$

305 where $Y = K_q Q_{22}$, $\mathcal{M} \in \mathcal{R}^{(p-m+q) \times (n-p)}$ is a tuning matrix.

306 *Example 2:* Consider the delay system

$$\begin{aligned} A &= \begin{bmatrix} 0 & 25 & -1 \\ 1 & 0 & 0 \\ -5 & 0 & 1 \end{bmatrix} & A_d &= \begin{bmatrix} 0.1 & 0.1 & 0 \\ 0 & 0.3 & -0.1 \\ 0 & 0.2 & 0 \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} & C &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \end{aligned} \quad (59)$$

307 from which

$$A_{11} = \begin{bmatrix} 0 & 25 \\ 1 & 0 \end{bmatrix} \quad A_{12} = \begin{bmatrix} -1 \\ 0 \end{bmatrix} \quad C_1 = [1 \quad 0].$$

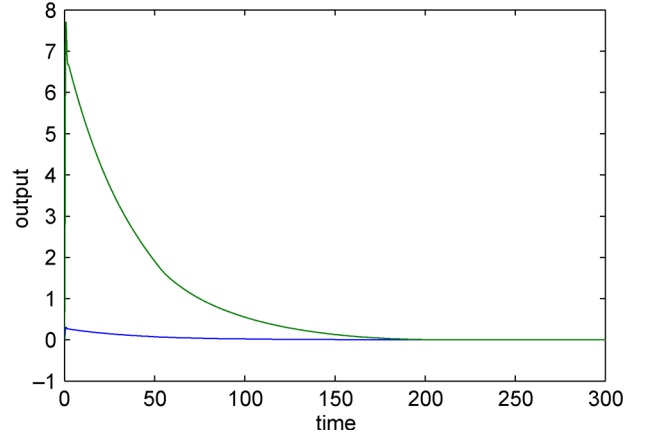


Fig. 3. Compensator-based controller design $h = 2.5s$.

It follows that

$$\lambda(A_{11} - A_{12}KC_1) = \pm \sqrt{(25 + K^2)}$$

and so (24) is infeasible. Now, consider designing a first-order
309 compensator. Choosing $D_2 = 1$, it follows that

$$\begin{aligned} A_q &= \begin{bmatrix} 0 & 25 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} & A_{qd} &= \begin{bmatrix} 0.1 & 0.1 & 0 \\ 0 & 0.3 & 0 \\ 0 & 0 & 0 \end{bmatrix} \\ B_q &= \begin{bmatrix} -1 & 0 \\ 0 & 0 \\ 1 & -1 \end{bmatrix} & B_{qd} &= \begin{bmatrix} -0.1 & 0 \\ 0 & 0 \\ 0 & 0 \end{bmatrix} \\ C_q &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}. \end{aligned}$$

Choosing $\delta = 50$, $\epsilon = 5$, $\mathcal{M} = [10 \quad 4]'$, and $d = 0$ (slowly
311 varying delay) with the maximum allowable delay $h = 0.25s$,
312 the LMITOOL solver returns

$$K_q = \begin{bmatrix} -25.38 & -0.37 \\ -25.32 & -5.03 \end{bmatrix}.$$

The augmented system with compensator given by

$$\begin{aligned} A_a &= \begin{bmatrix} H & DC \\ 0 & A \end{bmatrix} & A_{da} &= \begin{bmatrix} 0 & 0 \\ 0 & A_d \end{bmatrix} \\ B_a &= \begin{bmatrix} 0 \\ B \end{bmatrix} & C_a &= \begin{bmatrix} I_q & 0 \\ 0 & C \end{bmatrix} \end{aligned}$$

is asymptotically stabilized by the controller

$$[F_c \quad F] = [-0.369 \quad -25.38 \quad 1].$$

Taking the controller in the form of (10) and (11) where $\gamma = 10$,
316 $\rho = 10$. Simulation results with the switching gain and initial
317 conditions $[0.5, 0, 0]$, as shown in Figs. 3 and 4.

VII. CONCLUSION

A descriptor Lyapunov–Krasovskii functional method has
320 been introduced for SOF switching function design for systems
321

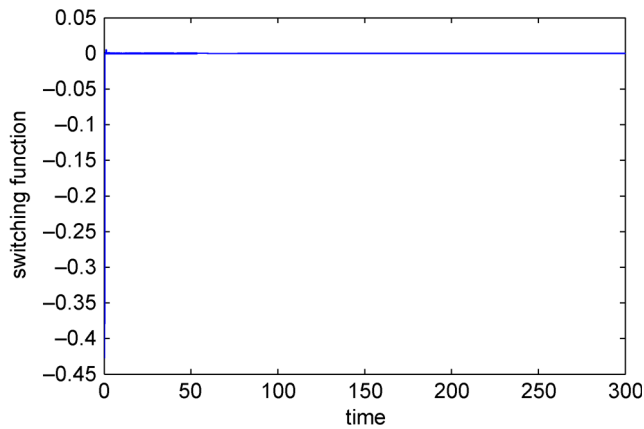


Fig. 4. Compensator-based controller design $h = 2.5s$.

322 with state time-varying delays. The delay is assumed bounded
 323 with a known upper bound, either slowly or fast varying. In
 324 addition, a novel stability analysis of the full-order closed-
 325 loop discontinuous time-delay system has been performed via
 326 the Krasovskii method, which is delay independent in $z_2(t)$
 327 (and thus the delay is restricted to be slowly varying) and
 328 delay dependent in $z_1(t)$, i.e., in the state of the reduced-order
 329 system. The proposed SOF design approach also applies to
 330 compensator-based design. Examples show the effectiveness of
 331 the method. For future work, the results can be extended to the
 332 interval delay case, where the lower bound on the delay is taken
 333 into account. The Razumikin approach can be employed for the
 334 stability analysis of the full-order closed-loop system with fast
 335 varying delay.

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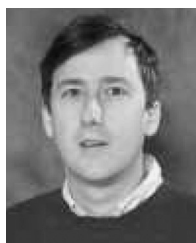
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On the Design of Sliding-Mode Static-Output-Feedback Controllers for Systems With State Delay

X. R. Han, Emilia Fridman, Sarah K. Spurgeon, and Chris Edwards

Abstract—This paper considers the development of sliding-mode-based output-feedback controllers for uncertain systems which are subject to time-varying state delays. A novel method is proposed for design of the switching surface. This method is based on the descriptor approach and leads to a solution in terms of linear matrix inequalities (LMIs). When compared to existing methods (even for systems without delays), the proposed method is efficient and less conservative than other results, giving a feasible solution when the Kimura–Davison conditions are not satisfied. No additional constraints are imposed on the dimensions or structure of the reduced order triple associated with design of the switching surface. The magnitude of the linear gain used to construct the controller is also verified as an appropriate solution to the reachability problem using LMIs. A stability analysis for the full-order time-delay system with discontinuous right-hand side is formulated. This paper facilitates the constructive design of sliding-mode static-output-feedback controllers for a rather general class of time-delay systems. A numerical example from the literature illustrates the efficiency of the proposed method.

Index Terms—Linear matrix inequalities (LMIs), sliding-mode control (SMC), static output feedback (SOF), time delay.

I. INTRODUCTION

SLIDING-MODE control (SMC) [1] is known for its complete robustness to so-called matched uncertainties (which can include time delays that satisfy matching conditions) and disturbances [2]–[4]. The control technique has been applied in many industrial areas [5]–[7]. Many early theoretical developments in SMC assume that all the system states are accessible. In the case where only a subset of states are measurable, which is relevant to a range of practical applications, either output feedback control or the observer-based method are required. Some work has considered implementation of SMC schemes using observers [8]–[10]. In [11], a sliding-mode observer has been shown to give a significant increase in performance in estimation of the unknown variables of a boost converter compared

to a traditional current-mode control strategy. A further interesting strand considers the fast output sampling method [12], [13]. Recently, in [14], a fast sampling method is employed for a discrete systems in the presence of time-varying delays where a sliding-mode controller is designed using linear matrix inequalities (LMIs) combined with a delta-operator approach. However, all these methods require additional computation. The most straightforward approach is to consider the study of SMC via static output feedback (SOF).

One problem of interest in the development of SMC via SOF is the design of the switching surface, which is effectively a reduced order SOF problem for a particular subsystem. Two different methods were proposed to design the sliding surface using eigenvalue assignment and eigenvector techniques in [15] and [16]. A canonical form was provided in [18] via which the SOFSMC design problem is routinely converted to an SOF stabilization problem. As stated in [20], all previous-reported methods for the existence problem are, in fact, equivalent to a particular SOF problem. The solution to the general SOF problem, even for linear time-invariant systems, is still open.

LMI methods have been considered within the context of sliding-mode controller design. For example, [21] and [22] presented LMIs methods to design static sliding-mode output-feedback controllers and [23] presented a necessary and sufficient condition to solve the existence problem in terms of LMIs for linear uncertain systems.

It is important to note that all the work described above does not consider an existence problem involving delay and many practical problems include such effects [24], [25]. In [26], the problem of the development of sliding-mode controllers for operation in the presence of single or multiple, constant or time-varying state delays has been solved. This uses the usual regular form method of solution and the uncertainty is assumed to be matched, where matched describes that uncertainty class which is implicit in the range of the input channels and will be rejected by an appropriately designed SMC strategy, although it is important to note that full state availability is assumed. This problem has also been considered in [27] where a class of uncertain time delay systems with multiple fixed delays in the system states is considered. This paper considers unmatched and time-varying parameter uncertainties, together with matched and bounded external disturbances, but again, full state information is assumed to be available to the controller. In [28], Lyapunov functionals were for the first time introduced for the analysis of time-varying delay. In [29], a descriptor

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85 approach to stability and control of linear systems with time-
86 varying delays, which is based on the Lyapunov–Krasovskii
87 techniques, was combined with results on the SMC of such
88 systems. The systems under consideration were subjected to
89 norm-bounded uncertainties and uncertain bounded delays and
90 the solution given in terms of LMIs. Reference [30] develops
91 a SMC synthesis for a class of uncertain time-delay systems
92 with nonlinear disturbances and unknown delay values whose
93 unperturbed dynamics is linear. The synthesis was based on a
94 new delay-dependent stability criterion. The controller is found
95 to be robust against sufficiently small delay variations and
96 external disturbances.

97 It is important to emphasize that much of the aforementioned
98 literature on SMC of time-delay systems assumes full-state
99 feedback. Reference [31] considered SMC of systems with
100 time-varying delay. This paper considered a solution via LMIs
101 for the existence problem. The current paper extends this contri-
102 bution to consider the solution of the existence and reachability
103 problems. Specifically, the selection of parameters for stability
104 of the full-order closed-loop system are obtained via LMIs. In
105 this paper, a compensator-based design problem is considered
106 using the proposed SOF approach. Example from the literature
107 illustrates the efficiency of the method. In Section II, the
108 problem formulation is described. The existence problem is
109 formulated in Section III. In Sections IV and V, stability of
110 the full-order closed-loop system is derived via LMIs, and the
111 reachability problem is presented. Compensator-based design
112 is demonstrated in Section VI.

113 II. PROBLEM FORMULATION

114 Consider an uncertain time-delay system

$$\begin{aligned} \dot{z}(t) &= Az(t) + A_d z(t - \tau(t)) + B(u(t) + \xi(t, z, u)) \\ y(t) &= Cz(t) \end{aligned} \quad (1)$$

115 where $z \in \mathcal{R}^n$, $u \in \mathcal{R}^m$, and $y \in \mathcal{R}^p$ with $m \leq p \leq n$. The
116 time-varying delay $\tau(t)$ is supposed to be bounded $0 \leq \tau(t) \leq$
117 h , and it may be either slowly varying (i.e., differentiable
118 delay with $\dot{\tau}(t) \leq d < 1$) or fast varying (piecewise continuous
119 delay). Assume that the nominal linear system (A, A_d, B, C)
120 is known and that the input and output matrices B and C are
121 both of full rank. The unknown function $\xi: \mathcal{R}_+ \times \mathcal{R}^n \times \mathcal{R}^m \rightarrow$
122 \mathcal{R}^n , which represents the system nonlinearities plus any model
123 uncertainties, is assumed to satisfy the matching condition and

$$\|\xi(t, z, u)\| < k_1 \|u\| + a(t, y) \quad (2)$$

124 for some known function $a: \mathcal{R}_+ \times \mathcal{R}^p \rightarrow \mathcal{R}_+$ and positive
125 constant $k_1 < 1$. It can be shown that if $\text{rank}(CB) = m$, there
126 exists a coordinate system in which the system (A, A_d, B, C)
127 has the structure

$$\begin{aligned} A &= \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} & A_d &= \begin{bmatrix} A_{d11} & A_{d12} \\ A_{d21} & A_{d22} \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ B_2 \end{bmatrix} & C &= [0 \quad T] \end{aligned} \quad (3)$$

where $B_2 \in \mathcal{R}^{m \times m}$ is nonsingular and $T \in \mathcal{R}^{p \times p}$ is orthogo- 128
nal. The system can be represented as 129

$$\begin{aligned} \dot{z}_1(t) &= A_{11}z_1(t) + A_{d11}z_1(t - \tau(t)) \\ &\quad + A_{12}z_2(t) + A_{d12}z_2(t - \tau(t)) \\ \dot{z}_2(t) &= \sum_{i=1}^2 (A_{2i}z_i(t) + A_{d2i}z_i(t - \tau(t))) \\ &\quad + B_2(u(t) + \xi(t, z, u)) \\ y(t) &= Cz(t). \end{aligned} \quad (4)$$

Consider the following switching function: 130 AQ1

$$S = \{z(t) \in \mathcal{R}^n : FCz(t) = 0\} \quad (5)$$

for some selected matrix $F \in \mathcal{R}^{m \times p}$ where by design 131
 $\det(FCB) \neq 0$. Let 132

$$[F_1 \quad F_2] = FT \quad (6)$$

where $F_1 \in \mathcal{R}^{p-m}$ and $F_2 \in \mathcal{R}^m$. As a result 133

$$FC = [F_1 C_1 \quad F_2] \quad (7)$$

where 134

$$C_1 = [0_{(p-m) \times (n-p)} \quad I_{(p-m)}]. \quad (8)$$

Therefore, $FCB = F_2 B_2$ and the square matrix F_2 is 135
nonsingular. By assumption, the uncertainty is matched, and 136
therefore the sliding motion is independent of the uncertainty 137
represented by $\xi(\cdot)$. In addition, because the canonical form in 138
(3), where it is necessary that the pair (A_{11}, A_{12}) is controllable 139
and (A_{11}, C_1) is observable, can be viewed as a special case 140
of the regular form normally used in sliding-mode controller 141
design, the switching function can also be expressed as 142

$$s(t) = z_2(t) + KC_1 z_1(t) \quad (9)$$

where $K \in \mathcal{R}^{m \times (p-m)}$ and is defined as $K = F_2^{-1} F_1$. 143

Then, a simple SMC law, depending on the output informa- 144
tion $Fy(t)$ can be defined by 145

$$u(t) = -\gamma Fy(t) - v(t) \quad (10)$$

where 146

$$v(t) = \begin{cases} \rho(t, y) \frac{Fy(t)}{\|Fy(t)\|}, & \text{if } Fy(t) \neq 0 \\ 0, & \text{otherwise} \end{cases} \quad (11)$$

where $\rho(t, y)$ is some positive scalar function of the outputs 147

$$\rho(t, y) = (k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2) / (1 - k_1)$$

where γ and γ_2 are positive design scalars [18]. The closed- 148
loops system (4) and (10) can be described by the following 149
equations: 150

$$\begin{aligned} \dot{z}_1(t) &= (A_{11} - A_{12}KC_1)z_1(t) \\ &\quad + (A_{d11} - A_{d12}KC_1)z_1(t - \tau(t)) \end{aligned}$$

$$\begin{aligned}
\dot{s}(t) &= (A_{21} - \gamma B_2 K C_1) z_1(t) \\
&+ (A_{d21} - \gamma B_2 K C_1) z_1(t - \tau(t)) \\
&+ (A_{22} - \gamma B_2) z_2(t) + (A_{d22} - \gamma B_2) z_2(t - \tau(t)) \\
&+ B_2 (\xi(t, z, u) - v(t)) \\
y(t) &= C z(t). \tag{12}
\end{aligned}$$

151

III. EXISTENCE PROBLEM

152 On the sliding manifold $s(t) = 0$, it is well known [19] that
 153 the reduced-order sliding motion is governed by a free motion
 154 with system matrix

$$\begin{aligned}
\dot{z}_1(t) &= (A_{11} - A_{12} K C_1) z_1(t) \\
&+ (A_{d11} - A_{d12} K C_1) z_1(t - \tau(t)). \tag{13}
\end{aligned}$$

155 Consider a Lyapunov–Krasovskii functional

$$\begin{aligned}
V(t) &= z_1^T(t) P z_1(t) + \int_{t-h}^t z_1^T(s) E z_1(s) ds \\
&+ \int_{t-\tau(t)}^t z_1^T(s) S z_1(s) ds \\
&+ h \int_{-h}^0 \int_{t+\theta}^t z_1^T(s) R \dot{z}_1(s) ds d\theta \tag{14}
\end{aligned}$$

156 where the symmetric matrices $P > 0$ and $E, S, R \geq 0$.

157 The condition $\dot{V}(t) < 0$ guarantees asymptotic stability of
 158 the reduced order system as in [32]. Differentiating $V(t)$
 159 along (13)

$$\begin{aligned}
\dot{V}(t) &= 2z_1^T(t) P \dot{z}_1(t) + h^2 \dot{z}_1^T(t) R \dot{z}_1(t) \\
&- h \int_{t-h}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds + z_1^T(t) (E + S) z_1(t) \\
&- z_1^T(t-h) E z_1(t-h) \\
&- (1 - \dot{\tau}(t)) z_1^T(t - \tau(t)) S z_1(t - \tau(t)). \tag{15}
\end{aligned}$$

160 Further using the identity

$$\begin{aligned}
-h \int_{t-h}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds &= -h \int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) R \dot{z}_1(s) ds \\
&- h \int_{t-\tau(t)}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds \tag{16}
\end{aligned}$$

161 and applying Jensen's inequality

$$\int_{t-\tau(t)}^t \dot{z}_1^T(s) R \dot{z}_1(s) ds \geq \frac{1}{h} \int_{t-\tau(t)}^t \dot{z}_1^T(s) ds R \int_{t-\tau(t)}^t \dot{z}_1(s) ds \tag{17}$$

$$\int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) R \dot{z}_1(s) ds \geq \frac{1}{h} \int_{t-h}^{t-\tau(t)} \dot{z}_1^T(s) ds R \int_{t-h}^{t-\tau(t)} \dot{z}_1(s) ds. \tag{18}$$

Then,

162

$$\begin{aligned}
\dot{V}(t) &\leq 2z_1^T(t) P \dot{z}_1(t) + h^2 \dot{z}_1^T(t) R \dot{z}_1(t) \\
&- (z_1(t) - z_1(t - \tau(t)))^T R (z_1(t) - z_1(t - \tau(t))) \\
&- (z_1(t - \tau(t)) - z_1(t - h))^T \\
&\times R (z_1(t - \tau(t)) - z_1(t - h)) \\
&+ z_1^T(t) (E + S) z_1(t) - z_1^T(t - h) E z_1(t - h) \\
&- (1 - d) z_1^T(t - \tau(t)) S z_1(t - \tau(t)). \tag{19}
\end{aligned}$$

Using the descriptor method as in [33] and the free-weighting
 163 matrices technique from [34], the right-hand side of the
 164 expression
 165

$$\begin{aligned}
0 &\equiv 2(z_1^T(t) P_2^T + \dot{z}_1^T(t) P_3^T) \\
&\times [-\dot{z}_1(t) + (A_{11} - A_{12} K C_1) z_1(t) \\
&+ (A_{d11} - A_{d12} K C_1) z_1(t - \tau(t))] \tag{20}
\end{aligned}$$

with matrix parameters $P_2, P_3 = \epsilon P_2 \in \mathcal{R}^{n-m}$ is
 166 added into the right-hand side of (19). Setting $\eta(t) =$
 167 $\text{col}\{z_1(t), \dot{z}_1(t), z_1(t - h), z_1(t - \tau(t))\}$, it follows that
 168

$$\dot{V}(t) \leq \eta^T(t) \Theta \eta(t) \leq 0 \tag{21}$$

if the matrix inequality

169

$$\Theta = \begin{bmatrix} \theta_{11} & \theta_{12} & 0 & \theta_{14} \\ * & \theta_{22} & 0 & \theta_{24} \\ * & * & \theta_{33} & \theta_{34} \\ * & * & * & \theta_{44} \end{bmatrix} < 0 \tag{22}$$

is feasible, where

170

$$\begin{aligned}
\theta_{11} &= P_2^T (A_{11} - A_{12} K C_1) \\
&+ (A_{11} - A_{12} K C_1)^T P_2 + E + S - R \\
\theta_{12} &= P - P_2^T + \epsilon (A_{11} - A_{12} K C_1)^T P_2 \\
\theta_{14} &= P_2^T (A_{d11} - A_{d12} K C_1) + R \\
\theta_{22} &= -\epsilon P_2 - \epsilon P_2^T + h^2 R \\
\theta_{24} &= \epsilon P_2^T (A_{d11} - A_{d12} K C_1) \\
\theta_{33} &= -(E + R) \\
\theta_{34} &= R \\
\theta_{44} &= -2R - (1 - d)S. \tag{23}
\end{aligned}$$

Multiplying matrix Θ from the right and the left by
 171 $\text{diag}\{P_2^{-1}, P_2^{-1}, P_2^{-1}, P_2^{-1}\}$ and its transpose, respectively, and
 172 denoting
 173

$$\begin{aligned}
Q_2 &= P_2^{-1} \quad \hat{P} = Q_2^T P Q_2 \quad \hat{R} = Q_2^T R Q_2 \\
\hat{E} &= Q_2^T E Q_2 \quad \hat{S} = Q_2^T S Q_2
\end{aligned}$$

174 it follows $\Theta < 0 \Leftrightarrow \hat{\Theta} < 0$, where

$$\hat{\Theta} = \begin{bmatrix} \hat{\theta}_{11} & \hat{\theta}_{12} & 0 & \hat{\theta}_{14} \\ * & \hat{\theta}_{22} & 0 & \hat{\theta}_{24} \\ * & * & \hat{\theta}_{33} & \hat{\theta}_{34} \\ * & * & * & \hat{\theta}_{44} \end{bmatrix} < 0 \quad (24)$$

$$\begin{aligned} \hat{\theta}_{11} &= (A_{11} - A_{12}KC_1)Q_2 \\ &\quad + Q_2^T(A_{11} - A_{12}KC_1)^T + \hat{E} + \hat{S} - \hat{R} \\ \hat{\theta}_{12} &= \hat{P} - Q_2 + \epsilon Q_2^T(A_{11} - A_{12}KC_1)^T \\ \hat{\theta}_{14} &= (A_{d11} - A_{d12}KC_1)Q_2 + \hat{R} \\ \hat{\theta}_{22} &= -\epsilon Q_2 - \epsilon Q_2^T + h^2 \hat{R} \\ \hat{\theta}_{24} &= \epsilon(A_{d11} - A_{d12}KC_1)Q_2 \\ \hat{\theta}_{33} &= -\hat{E} - \hat{R} \\ \hat{\theta}_{34} &= \hat{R} \\ \hat{\theta}_{44} &= -2\hat{R} - (1-d)\hat{S}. \end{aligned} \quad (25)$$

175 Define the variable Q_2 in the following form:

$$Q_2 = \begin{bmatrix} Q_{11} & Q_{12} \\ Q_{22}\mathcal{M} & \delta Q_{22} \end{bmatrix} \quad (26)$$

176 where Q_{22} is a $(p-m) \times (p-m)$ matrix, and $\mathcal{M} \in$
177 $\mathcal{R}^{(p-m) \times (n-p)}$ and $\delta \in \mathcal{R}$ are *a priori* selected tuning param-
178 eters. It follows that

$$KC_1Q_2 = [KQ_{22}\mathcal{M} \quad \delta KQ_{22}].$$

179 Defining

$$Y = KQ_{22}$$

180 it follows that

$$KC_1Q_2 = [Y\mathcal{M} \quad \delta Y]. \quad (27)$$

181 To construct K , substitute (27) into (25) to yield

$$\begin{aligned} \hat{\theta}_{11} &= A_{11}Q_2 - A_{12}[Y \quad \delta Y] + Q_2^T A_{11}^T \\ &\quad - [Y\mathcal{M} \quad \delta Y]^T A_{12}^T + \hat{E} + \hat{S} - \hat{R} \\ \hat{\theta}_{12} &= \hat{P} - Q_2 + \epsilon Q_2^T A_{11}^T - \epsilon [Y\mathcal{M} \quad \delta Y]^T A_{12}^T \\ \hat{\theta}_{14} &= A_{d11}Q_2 - A_{d12}[Y\mathcal{M} \quad \delta Y] + \hat{R} \\ \hat{\theta}_{22} &= -\epsilon Q_2 - \epsilon Q_2^T + h^2 \hat{R} \\ \hat{\theta}_{24} &= \epsilon A_{d11}Q_2 - \epsilon A_{d12}[Y\mathcal{M} \quad \delta Y] \\ \hat{\theta}_{33} &= -\hat{E} - \hat{R} \\ \hat{\theta}_{34} &= \hat{R} \\ \hat{\theta}_{44} &= -2\hat{R} - (1-d)\hat{S} \end{aligned} \quad (28)$$

182 with the tuning parameters δ , ϵ , and \mathcal{M} . The following Lemma
183 may now be stated.

184 *Lemma 1:* Given *a priori* selected tuning parameters ϵ , δ ,
185 and $\mathcal{M} \in \mathcal{R}^{(p-m) \times (n-p)}$, then (24) is an LMI in the decision
186 variables $\hat{P} > 0$, $\hat{E} \geq 0$, $\hat{S} \geq 0$, $\hat{R} \geq 0$ and matrices $Q_{22} \in$

$\mathcal{R}^{(p-m) \times (p-m)}$, $Q_{11} \in \mathcal{R}^{(n-p) \times (n-p)}$, $Q_{12} \in \mathcal{R}^{(n-p) \times (p-m)}$, 187
188 $Y \in \mathcal{R}^{m \times (p-m)}$. If a solution to (24) exists, which may be read-
189 ily obtained from available LMI tools, then the reduced order
190 system (13) is asymptotically stable for all differentiable delays
191 $0 \leq \tau(t) \leq h$, $\dot{\tau}(t) \leq d < 1$. Moreover, (13) is asymptotically
192 stable for all piecewise-continuous delays $0 \leq \tau(t) \leq h$, if the
193 LMI (24) is feasible with $\hat{S} = 0$.

Remark: The proposed method is suitable for SOF sliding- 194
195 mode controller design where Kimura–Davison conditions,
196 written as $n \leq m + p - 1$, are not satisfied as in [20]–[22]. No
197 constraints are imposed on the dimensions of the reduced-order
198 triple A_{11} , A_{12} , C_1 . This represents a constructive and efficient
199 approach to output-feedback-based design for a relatively broad
200 class of systems, which is less conservative than existing results
201 [20]–[22]; an example to illustrate the advantages of the method
202 for systems without time-delay is presented in [17].

IV. STABILITY OF THE FULL-ORDER 203 CLOSED-LOOP SYSTEM 204

It can be shown [18] that there exist a coordinate system in 205
206 which the system triple $(\bar{A}, \bar{A}_d, \bar{B}, \bar{F}\bar{C})$ has the property that

$$\begin{aligned} \bar{A} &= \begin{bmatrix} \bar{A}_{11} & \bar{A}_{12} \\ \bar{A}_{21} & \bar{A}_{22} \end{bmatrix} & \bar{A}_d &= \begin{bmatrix} \bar{A}_{d11} & \bar{A}_{d12} \\ \bar{A}_{d21} & \bar{A}_{d22} \end{bmatrix} \\ \bar{B} &= \begin{bmatrix} 0 \\ B_2 \end{bmatrix} & \bar{F}\bar{C} &= [0 \quad F_2] \end{aligned} \quad (29)$$

where $\bar{A}_{11} = A_{11} - A_{12}KC_1$ and $\bar{A}_{d11} = A_{d11} - A_{d12}KC_1$ 207
208 and F_2 is a design parameter. Let \bar{P} be a symmetric positive
209 definite matrix partitioned conformably with the matrices in
210 (29) so that

$$\bar{P} = \begin{bmatrix} \bar{P}_1 & 0 \\ 0 & \bar{P}_2 \end{bmatrix} \quad (30)$$

then the matrix \bar{P} satisfies the structural constraint 211

$$\bar{P}\bar{B} = \bar{C}^T F^T \quad (31)$$

if the design matrix $F_2 = B_2^T \bar{P}_2$. The matrix \bar{P} can be shown 212
213 to be a Lyapunov matrix for

$$\begin{aligned} \bar{A}_0 &= \bar{A} - \gamma \bar{B} \bar{F} \bar{C} \\ &= \bar{A} - \gamma \bar{B} [0 \quad F_2] \end{aligned} \quad (32)$$

for sufficiently large γ [18]. In the new coordinate system, the 214
215 uncertain system (1) can be written as

$$\dot{z}(t) = \bar{A}z(t) + \bar{A}_d z(t - \tau) + \bar{B}(u(t) + \xi(t, z, u)). \quad (33)$$

The closed-loop system will have the form 216

$$\dot{z}(t) = \bar{A}_0 z(t) + \bar{A}_d z(t - \tau) + \bar{B}(\xi(t, y_t) - v_y(t)). \quad (34)$$

For large enough $\gamma > 0$, these conditions are delay independent 217
218 with respect to the delay in z_2 .} However, for derivation of
219 this condition using Lyapunov–Krasovskii techniques, it is
220 necessary to consider the case where $\dot{\tau} \leq d < 1$. A stability

221 condition for the full-order closed-loop system can be derived
222 using the following Lyapunov–Krasovskii functional:

$$V(t) = z^T(t)\bar{P}z(t) + \int_{t-h}^t z^T(s)\bar{E}z(s)ds + \int_{t-\tau(t)}^t z^T(s)\bar{S}z(s)ds + h \int_{-h}^0 \int_{t+\theta}^t z^T(s)\bar{R}\dot{z}(s)dsd\theta \quad (35)$$

223 where the matrix \bar{E} , $\bar{S} \geq 0$ and $\bar{R} = \begin{bmatrix} \bar{R}_1 & 0 \\ 0 & 0 \end{bmatrix} \geq 0$ as it is
224 desired to determine a stability condition for the time delay
225 system which is delay independent with respect to delay in
226 $z_2(t)$. Differentiating $V(t)$ along the closed-loop trajectories

$$\begin{aligned} \dot{V}(t) &\leq 2z^T(t)\bar{P}\dot{z}(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) \\ &\quad - (z(t) - z(t - \tau(t)))^T \bar{R} (z(t) - (t - \tau(t))) \\ &\quad - (z(t - \tau(t)) - z(t - h))^T \\ &\quad \times \bar{R} (z(t - \tau(t)) - z(t - h)) \\ &\quad + z^T(t)(\bar{E} + \bar{S})z(t) - z^T(t - h)\bar{E}z(t - h) \\ &\quad - (1 - d)z^T(t - \tau(t))\bar{S}z(t - \tau(t)). \end{aligned} \quad (36)$$

227 Substitute the right-hand side of (34) into (36). Setting $\varsigma(t) =$
228 $col\{z(t), z(t - h), z(t - \tau(t))\}$, it follows that

$$\dot{V}(t) \leq \varsigma(t)^T \Phi_h \varsigma(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) + 2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) < 0 \quad (37)$$

229 is satisfied if $\varsigma^T(t)\Phi_h\varsigma(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) < 0$ and
230 $2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) < 0$, where

$$\Phi_h = \begin{bmatrix} \phi_{11} - \bar{R} & 0 & \bar{P}\bar{A}_d + \bar{R} \\ * & -(\bar{E} + \bar{R}) & \bar{R} \\ * & * & -2\bar{R} - (1 - d)\bar{S} \end{bmatrix} \quad (38)$$

231 with

$$\phi_{11} = \bar{A}_0^T \bar{P} + \bar{P} \bar{A}_0 + \bar{S} + \bar{E}. \quad (39)$$

232 Setting $\varrho(t) = col\{z(t), z(t - h), z(t - \tau), \xi(t, z, u) - v(t)\}$

$$\begin{aligned} &h^2\dot{z}^T(t)\bar{R}\dot{z}(t) \\ &= h^2 \left[z^T(t)\bar{A}_0 + z^T(t - \tau)\bar{A}_d^T \right. \\ &\quad \left. + (\xi(t, z, u) - v(t))^T \bar{B}^T \right] \bar{R} \\ &\quad \times [\bar{A}_0 z + \bar{A}_d z(t - \tau) + \bar{B}(\xi(t, z, u) - v(t))] \\ &= \varrho^T(t) \begin{bmatrix} \bar{A}_0^T \\ 0_{n_T} \\ \bar{A}_d^T \\ \bar{B}^T \end{bmatrix} \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} h^2 \bar{R}_1 \\ &\quad \times \begin{bmatrix} I_{(n-qm)} \\ 0 \end{bmatrix}^T \begin{bmatrix} \bar{A}_0^T \\ 0_{n_T} \\ \bar{A}_d^T \\ \bar{B}^T \end{bmatrix} \varrho(t). \end{aligned} \quad (40)$$

Using the Schur complement, $\xi^T(t)\Phi_h\xi(t) + h^2\dot{z}^T(t)\bar{R}\dot{z}(t) < 0$
holds if

$$\begin{bmatrix} h\bar{A}_0^T \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} \bar{R}_1 \\ \Phi_h \\ h\bar{A}_d^T \begin{bmatrix} I_{(n-m)} \\ 0 \end{bmatrix} \bar{R}_1 \\ *** \\ -\bar{R}_1 \end{bmatrix} < 0. \quad (41)$$

Inequality (41) is an LMI in the decision variables $\bar{P}_1 > 0$, $\bar{E} \geq 0$,
 $\bar{S} \geq 0$ and $\bar{R}_1 \geq 0$. Equation (37) is valid if (41) is satisfied
and given

$$\begin{aligned} &2z^T\bar{P}\bar{B}(\xi(t, z, u) - v(t)) \\ &= 2y^T F^T (\xi(t, z, u) - v(t)) \\ &\leq -2y^T F^T v(t) + 2\|Fy(t)\| \|\xi(t, z, u)\| \\ &= -2\rho(t, y) \|Fy(t)\| + 2\|Fy(t)\| \|\xi(t, z, u)\| \\ &< -2\|Fy(t)\| (\rho(t, y) - k_1 \|u(t)\| - \alpha(t, y)). \end{aligned} \quad (42)$$

However, by definition

$$\rho(t, y) = (k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2) / (1 - k_1)$$

and so by rearranging

$$\begin{aligned} \rho(t, y) &= k_1 \rho(t, y) + k_1 \gamma \|Fy(t)\| + \alpha(t, y) + \gamma_2 \\ &\geq k_1 (\|v(t)\| + \gamma \|Fy(t)\|) + \alpha(t, y) + \gamma_2 \\ &\geq k_1 \|u(t)\| + \alpha(t, y) + \gamma_2. \end{aligned} \quad (43)$$

From (37), if (41) is valid, then from (42) and (43),

$$\dot{V}(t) < -2\gamma_2 \|Fy(t)\| < 0 \quad \text{if } z(t) \neq 0 \quad (44)$$

and therefore the system is asymptotically stable.

Lemma 2: Given large enough γ , let there exist $n \times n$ ma-
trices $\bar{P}_1 > 0$, $\bar{E} \geq 0$, $\bar{S} \geq 0$, $\bar{R}_1 \geq 0$ from the LMI solver
such that LMI (41) holds. Given that the design parameters
 k_1 , $\alpha(t, y)$, γ_2 , and \bar{P}_2 have been selected so that (44) holds,
the closed-loop system (33) is asymptotically stable for all
differentiable delays $0 \leq \tau(t) \leq h$, $\dot{\tau}(t) \leq d \leq 1$.

V. FINITE-TIME REACHABILITY TO THE SLIDING MANIFOLD

Corollary: An ideal sliding motion takes place on the sur-
face S if

$$\|B_2^{-1} \bar{A}_0^L z(t)\| + \|B_2^{-1} \bar{A}_d^L z(t - \tau)\| < \gamma_2 - \eta \quad (45)$$

where the matrices \bar{A}_0^L and \bar{A}_d^L represent the last m rows of
 \bar{A}_0 and \bar{A}_d , respectively, and η is a small scalar satisfying
 $0 < \eta < \gamma_2$.

Proof:

$$\dot{s}(t) = F\bar{C}\bar{A}_0\bar{z}(t) + F\bar{C}\bar{A}_d\bar{z}(t - \tau) + F_2 B_2 (\xi(t, z, u) - v(t)). \quad (46)$$

256 Let $V_c : \mathcal{R}^m \rightarrow \mathcal{R}$ be defined by

$$V_c(s) = s^T(t) (F_2^{-1})^T \bar{P}_2 F_2^{-1} s(t). \quad (47)$$

257 Then, using the fact that $F_2^T = \bar{P}_2 B_2$ it follows that

$$\begin{aligned} (F_2^{-1})^T \bar{P}_2 F_2^{-1} F \bar{C} \bar{A}_0 &= B_2^{-1} \bar{A}_0^L \\ (F_2^{-1})^T \bar{P}_2 F_2^{-1} F \bar{C} \bar{A}_d &= B_2^{-1} \bar{A}_d^L. \end{aligned} \quad (48)$$

258 Then, it can be verified that

$$\begin{aligned} \dot{V}_c &= 2s^T(t) B_2^{-1} \bar{A}_0^L z(t) + 2s^T(t) B_2^{-1} \bar{A}_d^L z(t - \tau) \\ &\quad + 2s^T(t) (\xi(t, z, u) - v(t)) \\ &\leq 2 \|s(t)\| \|B_2^{-1} \bar{A}_0^L z(t)\| + 2 \|s(t)\| \\ &\quad \times \|B_2^{-1} \bar{A}_d^L z(t - \tau)\| - 2\gamma_2 \|s(t)\| \\ &< -2\eta \|s(t)\| \end{aligned} \quad (49)$$

259 if $z(t)$ and $z(t - \tau) \in \Omega$. It follows that there exists a t_0 such
260 that $z(t)$ and $z(t - \tau) \in \Omega$ for all $t > t_0$. Consequently, (49)
261 holds for all $t > t_0$. A sliding motion will thus be attained in
262 finite time.

263 *Example 1:* The following model of a liquid monopropel-
264 lant rocket motor has been considered in [35]. It is assumed
265 that the variable $\kappa = 0.8$ in this case, where $A_d(1, 1) = -\kappa$
266 and $A(1, 1) = \kappa - 1$. The outputs have been chosen to be the
267 second and fourth states so that in (1)

$$\begin{aligned} A &= \begin{bmatrix} -0.2 & 0 & 0 & 0 \\ 0 & 0 & 0 & -1 \\ -1 & 0 & -1 & 1 \\ 0 & 1 & 1 & 0 \end{bmatrix} & A_d &= \begin{bmatrix} -0.8 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ 1 \\ 0 \\ 0 \end{bmatrix} & C &= \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}. \end{aligned} \quad (50)$$

268 Clearly, the Kimura–Davison conditions are not met. Here,
269 the rate at which the delay varies with time has been examined
270 with $d = 0$, a slow varying delay. The gain from the LMI tool
271 solver with $\delta = 50$, $\epsilon = 0.5$, $h = 0.45s$, $\bar{P}_2 = 1$, $\gamma = 2.9$, and
272 $M = [5 \ 0.2]$, yields $F = [-1 \ -2.0754]$. The poles of the
273 sliding-mode dynamics are $\{-2.67, -0.4, -0.2\}$. A simu-
274 lation was performed with the initial state values $[1 \ 1 \ 1 \ 1]$.
275 As shown, the LMI solver gave a feasible result for stability
276 for $h \leq 0.45s$, but in simulation, the closed-loop system only
277 became unstable for $h \geq 1s$ (Fig. 1); this is due to the conserv-
278 ativeness of the method. The LMI solver gave feasible closed-
279 loop stability results for controller gain $\gamma \geq 2.9$ while a choice
280 of $\gamma \geq 1$ in simulation was able to stabilize the system with the
281 compensation of longer settling time. The switching function
282 for $h = 0.45s$, $\gamma = 2.9$ is shown in Fig. 2.

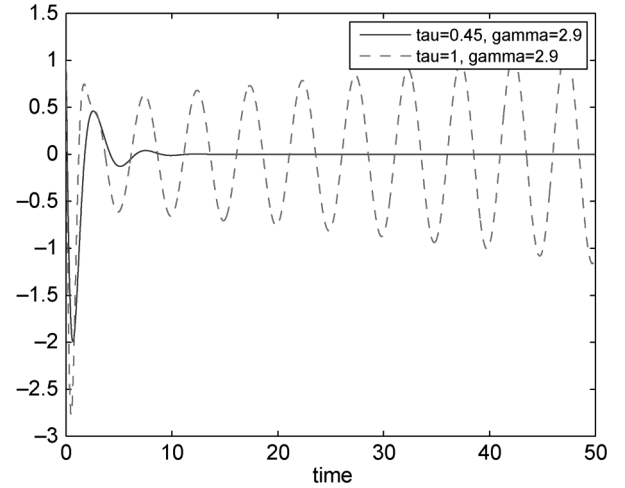


Fig. 1. Output against time.

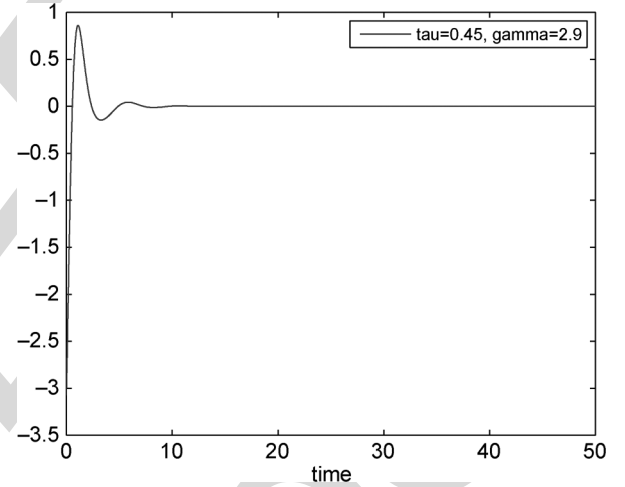


Fig. 2. Switching function.

VI. COMPENSATOR-BASED EXISTENCE PROBLEM 283

For certain system triples (A_{11}, A_{12}, C_1) , LMI (24) is known
284 to be infeasible. In this case, consider a dynamic compensator
285 similar to that of El-Khazali and DeCarlo [36]
286

$$\dot{z}_c(t) = H z_c(t) + D y(t) \quad (51)$$

where the matrices $H \in \mathcal{R}^{q \times q}$ and $D \in \mathcal{R}^{q \times p}$ are to be deter-
287 mined. Define a new hyperplane in the augmented state space,
288 formed from the plant and compensator state spaces, as
289

$$S_c = \{(z(t), z_c(t)) \in \mathcal{R}^{n+q} : F_c z_c(t) + F z(t) = 0\} \quad (52)$$

where $F_c \in \mathcal{R}^{m \times q}$ and $F \in \mathcal{R}^{m \times p}$. Define $D_1 \in \mathcal{R}^{q \times (p-m)}$
290 and $D_2 \in \mathcal{R}^{q \times m}$ as
291

$$[D_1 \ D_2] = DT \quad (53)$$

then the compensator can be written as
292

$$\dot{z}_c(t) = H z_c(t) + D_1 C_1 z_1(t) + D_2 z_2(t) \quad (54)$$

293 where C_1 is defined in (8). The sliding motion, obtained by
294 eliminating the coordinates $z_2(t)$, can be written as

$$\begin{aligned}\dot{z}_1(t) &= (A_{11} - A_{12}KC_1)z_1(t) - A_{12}K_c z_c(t) \\ &\quad + (A_{d11} - A_{d12}KC_1)z_1(t - \tau) - A_{d12}K_c z_c(t - \tau) \\ \dot{z}_c(t) &= (D_1 - D_2K)C_1 z_1(t) + (H - D_2K_c)z_c(t)\end{aligned}\quad (55)$$

295 where $K = F_2^{-1}F_1$ and $K_c = F_2^{-1}F_c$, then similar to [37], the
296 design problem becomes one of selecting a compensator, re-
297 presented by the matrices D_1 , D_2 , and H , and a hyperplane,
298 represented by the matrices K and K_c , so that the system

$$\begin{aligned}\begin{bmatrix} \dot{z}_1(t) \\ \dot{z}_c(t) \end{bmatrix} &= \underbrace{\begin{bmatrix} A_{11} - A_{12}KC_1 & -A_{12}K_c \\ (D_1 - D_2K)C_1 & H - D_2K_c \end{bmatrix}}_{A_c} \begin{bmatrix} z_1(t) \\ z_c(t) \end{bmatrix} \\ &\quad + \underbrace{\begin{bmatrix} A_{d11} - A_{d12}KC_1 & -A_{d12}K_c \\ 0 & 0 \end{bmatrix}}_{A_{cd}} \begin{bmatrix} z_1(t - \tau) \\ z_c(t - \tau) \end{bmatrix}\end{aligned}\quad (56)$$

299 is stable. To obtain the compensator gains this problem can be
300 shown to be a new output-feedback problem with

$$\begin{aligned}A_c &= \underbrace{\begin{bmatrix} A_{11} & 0 \\ 0 & 0 \end{bmatrix}}_{A_q} - \underbrace{\begin{bmatrix} A_{12} & 0 \\ D_2 & -I_q \end{bmatrix}}_{B_q} \underbrace{\begin{bmatrix} K & K_c \\ D_1 & H \end{bmatrix}}_{K_q} \underbrace{\begin{bmatrix} C_1 & 0 \\ 0 & I_q \end{bmatrix}}_{C_q} \\ A_{cd} &= \underbrace{\begin{bmatrix} A_{d11} & 0 \\ 0 & 0 \end{bmatrix}}_{A_{qd}} - \underbrace{\begin{bmatrix} A_{d12} & 0 \\ 0 & 0 \end{bmatrix}}_{B_{qd}} \underbrace{\begin{bmatrix} K & K_c \\ D_1 & H \end{bmatrix}}_{K_q} \underbrace{\begin{bmatrix} C_1 & 0 \\ 0 & I_q \end{bmatrix}}_{C_q}.\end{aligned}\quad (57)$$

301 The existence problem represented by system (56), where A_c
302 and A_{cd} are partitioned as in (57) and D_2 is a tuning parameter,
303 can be solved as for the noncompensated case (13). Similarly
304 to (27),

$$\begin{aligned}K_q C_q Q_2 &= K_q \begin{bmatrix} 0_{(p-m+q) \times (n-p)} & I_{p-m+q} \end{bmatrix} \begin{bmatrix} Q_{11} & Q_{12} \\ Q_{22} \mathcal{M} & \delta Q_{22} \end{bmatrix} \\ &= [K_q Q_{22} \mathcal{M} \quad \delta K_q Q_{22}] \\ &= [Y \mathcal{M} \quad \delta Y]\end{aligned}\quad (58)$$

305 where $Y = K_q Q_{22}$, $\mathcal{M} \in \mathcal{R}^{(p-m+q) \times (n-p)}$ is a tuning matrix.

306 *Example 2:* Consider the delay system

$$\begin{aligned}A &= \begin{bmatrix} 0 & 25 & -1 \\ 1 & 0 & 0 \\ -5 & 0 & 1 \end{bmatrix} & A_d &= \begin{bmatrix} 0.1 & 0.1 & 0 \\ 0 & 0.3 & -0.1 \\ 0 & 0.2 & 0 \end{bmatrix} \\ B &= \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} & C &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}\end{aligned}\quad (59)$$

307 from which

$$A_{11} = \begin{bmatrix} 0 & 25 \\ 1 & 0 \end{bmatrix} \quad A_{12} = \begin{bmatrix} -1 \\ 0 \end{bmatrix} \quad C_1 = [1 \quad 0].$$

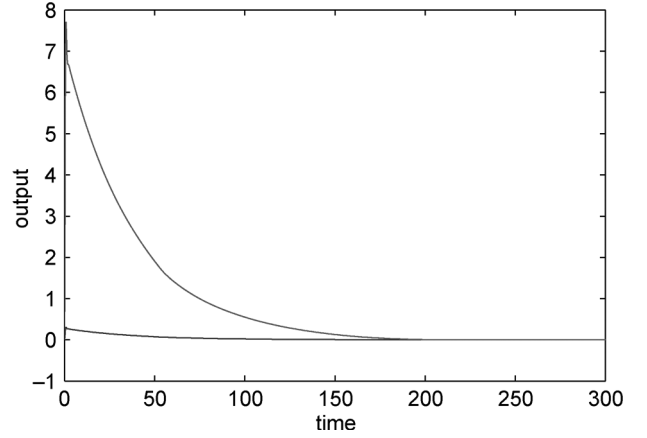


Fig. 3. Compensator-based controller design $h = 2.5s$.

It follows that

$$\lambda(A_{11} - A_{12}KC_1) = \pm \sqrt{(25 + K^2)}$$

and so (24) is infeasible. Now, consider designing a first-order
309 compensator. Choosing $D_2 = 1$, it follows that

$$\begin{aligned}A_q &= \begin{bmatrix} 0 & 25 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} & A_{qd} &= \begin{bmatrix} 0.1 & 0.1 & 0 \\ 0 & 0.3 & 0 \\ 0 & 0 & 0 \end{bmatrix} \\ B_q &= \begin{bmatrix} -1 & 0 \\ 0 & 0 \\ 1 & -1 \end{bmatrix} & B_{qd} &= \begin{bmatrix} -0.1 & 0 \\ 0 & 0 \\ 0 & 0 \end{bmatrix} \\ C_q &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}.\end{aligned}$$

Choosing $\delta = 50$, $\epsilon = 5$, $\mathcal{M} = [10 \quad 4]'$, and $d = 0$ (slowly
311 varying delay) with the maximum allowable delay $h = 0.25s$,
312 the LMITOOL solver returns

$$K_q = \begin{bmatrix} -25.38 & -0.37 \\ -25.32 & -5.03 \end{bmatrix}.$$

The augmented system with compensator given by

$$\begin{aligned}A_a &= \begin{bmatrix} H & DC \\ 0 & A \end{bmatrix} & A_{da} &= \begin{bmatrix} 0 & 0 \\ 0 & A_d \end{bmatrix} \\ B_a &= \begin{bmatrix} 0 \\ B \end{bmatrix} & C_a &= \begin{bmatrix} I_q & 0 \\ 0 & C \end{bmatrix}\end{aligned}$$

is asymptotically stabilized by the controller

$$[F_c \quad F] = [-0.369 \quad -25.38 \quad 1].$$

Taking the controller in the form of (10) and (11) where $\gamma = 10$,
316 $\rho = 10$. Simulation results with the switching gain and initial
317 conditions $[0.5, 0, 0]$, as shown in Figs. 3 and 4.

VII. CONCLUSION

A descriptor Lyapunov–Krasovskii functional method has
320 been introduced for SOF switching function design for systems
321

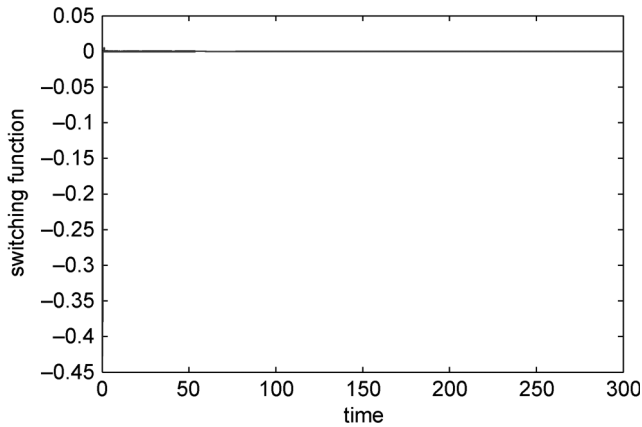


Fig. 4. Compensator-based controller design $h = 2.5s$.

322 with state time-varying delays. The delay is assumed bounded
 323 with a known upper bound, either slowly or fast varying. In
 324 addition, a novel stability analysis of the full-order closed-
 325 loop discontinuous time-delay system has been performed via
 326 the Krasovskii method, which is delay independent in $z_2(t)$
 327 (and thus the delay is restricted to be slowly varying) and
 328 delay dependent in $z_1(t)$, i.e., in the state of the reduced-order
 329 system. The proposed SOF design approach also applies to
 330 compensator-based design. Examples show the effectiveness of
 331 the method. For future work, the results can be extended to the
 332 interval delay case, where the lower bound on the delay is taken
 333 into account. The Razumikin approach can be employed for the
 334 stability analysis of the full-order closed-loop system with fast
 335 varying delay.

336

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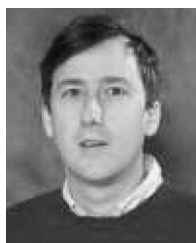
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