**Linear Stability Analysis of Lid Driven and Convective Flows** 

**Accelerated by an Efficient Fully Coupled Time-Marching Algorithm** 

#### Yu. Feldman and A. Gelfgat

School of Mechanical Engineering Faculty of Engineering Tel-Aviv University

Third International Symposium on Instabilities and Bifurcations in Fluid Dynamics, Nottingham, Great Britain, August 2009.

## Outline

Pressure-velocity coupled formulation of the Navier-Stokes equations
The direct and the multigrid approach
Numerical technique
Analytical Solution Accelerated (ASA) smoother
Comparison to existing benchmark solutions
Application to the linear stability analysis

≻Conclusions

Incompressible N-S Equations – Numerical Challenge

Continuity -  $\nabla \cdot \boldsymbol{u} = 0$ 

Momentum-

$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \nabla)\boldsymbol{u} = -\nabla p + \frac{1}{\operatorname{Re}} \nabla^2 \boldsymbol{u}$$

No separate equation for pressureNo boundary conditions for pressure

Incompressible N-S Equations – Numerical Challenge (Cont.)

Pressure Projection Approach

- ✓ High numerical robustness
- ✓ Low memory consumption
- Slow rate of numerical convergence
- X Non-physical pressure field
- Not applicable for liquid solid interface problems

Pressure–Velocity Coupled Approach

- ✓ High rate of numerical convergence
- ✓ The "most natural " way to solve N-S equations
- $\checkmark$  The obtained pressure is physical
- **×** High memory consumption
- Not as numerically robust as pressure projection methods

# **Benchmark Problems**

Lid-Driven Cubic Cavity



$$\nabla \cdot \boldsymbol{u} = 0$$
  

$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \nabla) \boldsymbol{u} = -\nabla p + \frac{1}{\text{Re}} \nabla^2 \boldsymbol{u}$$
  

$$\checkmark \text{Explicit Discretization}$$
  

$$(\boldsymbol{u}^n \cdot \nabla) \boldsymbol{u}^n$$

Semi-Implicit Discretization

$$(\boldsymbol{u}^n\cdot\nabla)\boldsymbol{u}^{n+1}$$

**Realistic Boundary Conditions:** 

u = 0 - at all static walls no slip/no penetration

|u| = v -at the moving wall the flow velocity is equal to that of the moving wall itself No boundary condition for pressure is needed

## **Benchmark Problems (Cont.)**

Differently Heated Rectangular and Cubic Cavity (Boussinesq Approximation)



$$\nabla \cdot \boldsymbol{u} = 0$$
  
$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \nabla) \boldsymbol{u} = -\nabla p + \sqrt{\frac{1}{\mathrm{Gr}}} \nabla^2 \boldsymbol{u} + \theta \vec{\boldsymbol{e}_z}$$
  
$$\frac{\partial \theta}{\partial t} + (\boldsymbol{u} \cdot \nabla) \theta = \frac{1}{\mathrm{Pr}\sqrt{\mathrm{Gr}}} \nabla^2 \theta$$

✓ Explicit Discretization

$$(\boldsymbol{u}^n\cdot\nabla)\boldsymbol{u}^n \quad (\boldsymbol{u}^n\cdot\nabla)\boldsymbol{\theta}^{n+1}$$

Semi-Implicit Discretization

$$(\boldsymbol{u}^n\cdot\nabla)\boldsymbol{u}^{n+1}$$
  $(\boldsymbol{u}^n\cdot\nabla)\boldsymbol{\theta}^{n+1}$ 

**Boundary Conditions:** 

 $\theta = 1, \theta = 0$  -isothermal vertical walls, v = H/Wv = 0

$$u = 0$$
 -at all walls,

No boundary condition for pressure is needed

 $\frac{\partial \theta}{\partial n} = 0 \text{ or } \theta = 1 - y \quad \text{-horizontal and}$ 

6/27

lateral walls

## Time and spatial discretization

Second order backward differentiation -  $\frac{\partial f^{n+1}}{\partial t} = \frac{3f^{n+1} - 4f^n + f^{n-1}}{2\Delta t} + O(\Delta t^2)$ Stokes operator linearization 
— Temperature – velocity decoupling  $\left(a_{(i,j,k)}^{\theta} - \frac{3}{2\Lambda\tau}\right)\theta_{(i,j,k)}^{n+1} + \sum_{i,j,k}a_{i,j,k}^{\theta}\theta_{i,j,k}^{n+1} = RHP_{\theta}^{n}$ Energy -Continuity -  $\frac{\left(u_{(i,j,k)}^{n+1} - u_{(i-1,j,k)}^{n+1}\right)}{Hx(i-1)} + \frac{\left(v_{(i,j,k)}^{n+1} - v_{(i,j-1,k)}^{n+1}\right)}{Hv(i-1)} + \frac{\left(w_{(i,j,k)}^{n+1} - w_{(i,j,k-1)}^{n+1}\right)}{Hz(k-1)} = 0$ Momentum-  $\left(a_{(i,j,k)}^{u} - \frac{3}{2\Lambda\tau}\right)u_{(i,j,k)}^{n+1} + \sum_{(i,j,k)}a_{(i,j,k)}^{u}u_{(i,j,k)}^{n+1} - \nabla p^{(n+1)} = RHP_{u}^{n}$ 

Conservative second order control volume method



#### The Multigrid Algorithm Symmetrical Coupled Gauss-Seidel Smoothing Operator (SCGS)

S.P. Vanka (1985) – analytical solution for a *single* finite volume



$$(u,v)^{\text{new}} = (u,v)^{\text{old}} + r_{(u,v)}(u,v)'$$

$$p^{\text{new}} = p^{\text{old}} + r_p p'$$

$$A_1 = a_e^u - \frac{3}{2\Delta\tau} \qquad A_3 = a_w^u - \frac{3}{2\Delta\tau}$$

$$A_5 = a_n^u - \frac{3}{2\Delta\tau} \qquad A_9 = a_s^u - \frac{3}{2\Delta\tau}$$

for Stokes operator and constant time step formulation  $A_1, A_3, A_5, A_9$ are constants

# Accelerated Coupled Line Gauss-Seidel Smoother (ASA-CLGS) -2D

Zeng and Wesseling (1993) – CLGS: Horizontal (vertical) sweeping with horizontally (vertically) adjacent pressure linkage Feldman and Gelfgat (2008) – ASA-CLGS:Horizontal (vertical) sweeping without horizontally (vertically) adjacent pressure linkage



## Accelerated Coupled Line Gauss-Seidel Smoother (ASA-CLGS) -2D, (Cont)

Zeng and Wesseling (1993) – CLGS:

Feldman and Gelfgat (2008) – ASA-CLGS:

 $\begin{aligned} A_{i+1/2,j}^{(x)} u'_{i+1/2,j} + B_{i+1/2,j}^{(x)} p'_{i,j} &= R_{i+1/2,j}^{(x)} & A_{i+1/2,j}^{(x)} u'_{i+1/2,j} + B_{i+1/2,j}^{(x)} p'_{i,j} &= R_{i+1/2,j}^{(x)} \\ A_{i-1/2,j}^{(x)} u'_{i-1/2,j} - B_{i-1/2,j}^{(x)} p'_{i,j} &= R_{i-1/2,j}^{(x)} & A_{i-1/2,j}^{(x)} u'_{i-1/2,j} - B_{i-1/2,j}^{(x)} p'_{i,j} &= R_{i-1/2,j}^{(x)} \\ A_{i,j+1/2}^{(y)} v'_{ij+1/2} - B_{i,j+1/2}^{(y)} \left( p'_{i,j+1} - p'_{i,j} \right) &= R_{i,j+1/2}^{(y)} & A_{i,j+1/2}^{(y)} v'_{ij+1/2} + B_{i,j+1/2}^{(y)} p'_{i,j} &= \widetilde{R}_{i,j+1/2}^{(y)} \\ A_{i,j}^{(x)} \left( u'_{i+1/2,j} - u'_{i-1/2,j} \right) + A_{i,j}^{(y)} \left( v'_{i,j+1/2} - v'_{i,j-1/2} \right) &= 0 & A_{i,j}^{(x)} \left( u'_{i+1/2,j} - u'_{i-1/2,j} \right) + A_{i,j}^{(y)} \left( v'_{i,j+1/2} - v'_{i,j-1/2} \right) &= 0 \end{aligned}$ 

where

re 
$$\widetilde{R}_{i,j+1/2}^{(y)} = R_{i,j+1/2}^{(y)} + B_{i,j+1/2}^{(y)} p'_{i,j+1/2}$$

# A Schematic Description of ASA-CLGS Smoother



# CLGS and ASA-CLGS Efficiency Estimation for 2D





### Advantages of ASA-CLGS Approach

Zeng and Wesseling (CLGS, 1993) Feldman and Gelfgat (ASA-CLGS, 2008)

☑ Still effective for stretched grids.

**Still effective for flows with a dominating direction** 

- **\*** Block three-diagonal system is to be solved numerically.
- Increased number of arithmetic operations when transferring from 2D to 3D geometry
- ☑ There exists an analytical solution for the entire corrections row (column ).
- ✓ Only O(5M) operations are needed to obtain the entire row (column) corrections per one sweep for both 2D and 3D geometries

## The Multigrid Characteristics



☑ Approximately O(N) of the CPU memory and time consumption for both 2D and 3D configurations **FPCD** and Multigrid Approaches – Pros and Cons

FPCD

### Multigrid

✓Independent of operating conditions :
 ∆t magnitude, Re, Gr numbers
 ✓Good Scalability of LU decomposition

- **☑** Very effective for small time steps.
- ✓ Very good scalability for both 2D and 3D configurations.
- **☑** Small memory consumption.
- Extremely memory demanding for 3D × Performance of the method depends
   (3D calculations is still a challenge) on operating conditions
- Still insufficient scalability of back substitution process
- Performnce of the method depends on initial guess

**FPCD** and Multigrid Approaches – Pros and Cons (Cont)

CPU time consumption for one time step -2D configuration



#### Cubic lid- driven cavity, grid resolution 103<sup>3</sup> Comparison with Albensoeder & Kuhlmann, 2005. Flow at Re= 1000



#### **Cubic lid-driven cavity, grid resolution 103<sup>3</sup> (cont)** Comparison with Albensoeder & Kuhlmann, 2005. Flow at Re= 1000



#### Cubic lid-driven cavity, grid resolution 103<sup>3</sup> (Cont.2) Comparison with Experiments of A. Liberzon, 2008. Flow at Re= 1000



**Subproject :** which resolution is necessary to fit experimental data with larger Reynolds number ?



### Temperature Distribution in a Laterally Heated Cubic Cavity, 103<sup>3</sup> Nodes

flow at  $Ra=10^6$  $\theta$  $\theta$ 1 1 0.9 0.9 0.8 0.8 0.7 0.7 0.6 0.6 0.5 0.5 0.4 0.4 0.3 0.3 0.2 0.2 0.1 0.1 0 0 23/27



## **Application to the Stability Analysis**

Newton iteration for steady state solutionL.S. Tuckerman 1999 $(N_U + L)u = (N + L)U$  $U \leftarrow U - u$  $\left[ (I - \Delta tL)^{-1} (I + \Delta tN_U - I) \right] u = \left[ (I - \Delta tL)^{-1} (I + \Delta tN(U) - I) \right] U$ Difference between two<br/>consecutive linearizedDifference between two<br/>to state the state solution

time steps

consecutive time steps

For large  $\Delta t (I - \Delta t L)^{-1} \approx L^{-1}$ is a preconditioner for  $N_U + L$  25/27

# Application to the Stability Analysis (Cont)

Inverse power method for the leading eigen value

L.S. Tuckerman 1999

$$u_{n+1} = \left(N_U + L\right)^{-1} u_n$$

$$\left[\left(I - \Delta tL\right)^{-1} \left(I + \Delta tN_U - I\right)\right] u_{n+1} = \underbrace{\left(I - \Delta tL\right)^{-1}}_{-1} \Delta tu_n$$

Difference between two consecutive linearized time steps

Difference between two consecutive time steps of the Stokes operator

Good performance for 2D configuration Still a challenge for 3D configuration

#### Conclusions

- An Accelerated Semi-Analytical Coupled Line Implicit Gauss-Seidel Smoother (ASA-CLGS) and Full Pressure Coupled Direct Solution (FPDS) were developed and implemented for the solution of incompressibel N-S equations.
- ✓ The Navier-Stokes and Boussinesq equations are solved without pressure-velocity decoupling.
- ✓ The code was verified on existing benchmark solutions for the liddriven and thermally driven cavities.
- ✓ The potential implementation of the developed time marching solvers to the linear stability analysis was studied.
- ✓ The characteristic CPU times consumed for a single time step per one node and per one CPU are of order 5 ×10<sup>-3</sup> msec and 10<sup>-2</sup> msec for 2D and 3D calculations, respectively.